

Internal Assessment Test – I Scheme and Solutions

Sub:	Fluid Mechanics					Code:	15ME44		
Date:	28/03/2017	Duration:	90 mins	Max Marks:	50	Sem:	IV	Branch:	Mechanical

PART – A (Answer any one full question)

Marks

Marks Split up

- 1 a) Distinguish between
i) Mass density and specific weight

[06]

2 Marks

Mass density	Specific weight
<ul style="list-style-type: none"> Mass density is defined as the ratio of mass of the fluid to the Volume of fluid $\rho = \frac{m}{V} \cdot \text{Kg/m}^3$	<ul style="list-style-type: none"> Specific weight is defined as ratio of weight of the liquid to the Volume of the liquid $w = \frac{W}{V} = \frac{(m \times g)}{V} = \rho g$ $w = \rho g$

- ii) Newtonian Fluid and Non-Newtonian Fluid

2 Marks

Newtonian fluid	Non-Newtonian fluid
<ul style="list-style-type: none"> The fluids which obey's law Newton law of Viscosity is called Newtonian fluid Shear stress will be directly proportional to the rate of shear strain 	<ul style="list-style-type: none"> The fluids which doesn't obey law of Viscosity is called Non-newtonian fluid. Shear stress (τ) will not be directly proportional to the rate of shear strain

- iii) Absolute Viscosity and Kinematic Viscosity

2 Marks

Absolute Viscosity	Kinematic Viscosity
<ul style="list-style-type: none"> It is defined as the resistance offered the fluid from one layer to other layer Unit = Ns/m^2 	<ul style="list-style-type: none"> It is defined as the ratio of Absolute Viscosity to the density of fluid $\nu = \frac{\mu}{\rho} = \text{m}^2/\text{s}$

b) Explain the effect of variation of temperature on viscosity of liquid and gases [04]

Variation of Viscosity in Liquid →

When the temperature ^{of the liquid.} increases, Viscosity of the liquid decreases.

temperature $\propto \frac{1}{\text{viscosity}}$

• This is mainly due to the Cohesive forces in the liquid particles. As the temperature increases cohesive force decrease hence Viscosity also decreases.

Variation of Viscosity in Gas →

When the temperature of liquid increases, the Viscosity of gases also increases.

• This is mainly due to the molecular momentum of the gases.

When the temperature increases, the Kinetic Energy of gas molecules also increases hence molecular momentum also increases.

∴ ~~Kinematic~~ Viscosity also increases.

2 Marks

2 Marks

2 a) Differentiate between

i) Steady and Unsteady flow

Steady flow is one in which the properties of the fluid do not vary with time. i.e., $\frac{dv}{dt} = 0$.
While an unsteady flow is one in which the properties of the fluid vary with time. i.e., $\frac{dv}{dt} \neq 0$.

ii) Uniform and non-uniform flow

Uniform flow is one in which the properties of the fluid do not vary with the surface. i.e., $\frac{dv}{ds} = 0$.
While a non-uniform flow is one in which properties of the fluid varies with respect to the surface. i.e., $\frac{dv}{ds} \neq 0$.

iii) Laminar and Turbulent flow

Laminar flow is one in which the fluid flows in a specified path. i.e., the streamlines flow parallel to each other & in a specific direction.
While Turbulent flow is one where the fluid flows in zig-zag direction ~~since~~ since the particles flow in an zig-zag direction.

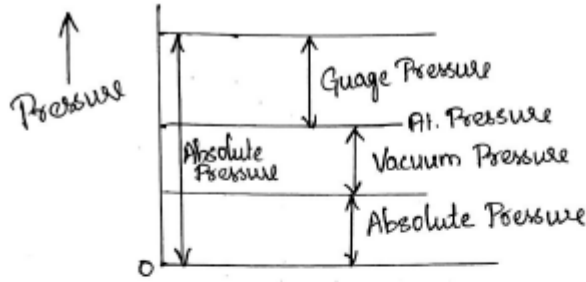
2 Marks

2 Marks

2 Marks

[06]

b) Establish a relationship among absolute, gauge and atmospheric pressure with sketch [04]



2 Marks

Gauge Pressure is measured above the atmospheric pressure.

2 Marks

Vacuum Pressure is the pressure measured below the atmospheric pressure.

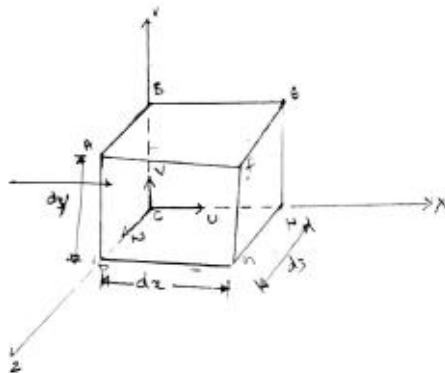
Absolute Pressure: It is the pressure measured from \emptyset pressure. zero pressure

$$\therefore P_{abs} = P_{atm} + P_{gauge}$$

$$P_{abs} = P_{atm} - P_{vacuum}$$

PART - B (Answer any two questions)

3 Derive with usual notations, the continuity equation for 3D flow and modify the equation for steady and incompressible flow. [10]



3 Marks

Consider a fluid Element whose dimensions are dx, dy and dz along $x, y, \& z$ axes respectively.

let $u, v, \& w$ be the velocities in x, y and z direction.

The mass/time of fluid entering the surface ABCD along xx -direction is given by

$$\dot{m}_{abcd} = \int x \, dy \, dz \times u \quad \text{--- (1)}$$

It's observed that, the density and velocity changes when it travels through the surface

②- Density changes by $\int + \frac{\partial \rho}{\partial x} \cdot dx$

Velocity changes

$$u + \frac{\partial u}{\partial x} \, dx$$

$$\therefore \dot{m}_{\text{efflux}} = \int \left(\rho + \frac{\partial \rho}{\partial z} dz + u + \frac{\partial u}{\partial z} dz \right) dy dz$$

$$= dx \left[\rho + \frac{\partial \rho}{\partial z} dz + u + \frac{\partial u}{\partial z} dz \right] dy dz$$

$$= dx \left[\rho + u + \frac{\partial(\rho u)}{\partial z} dz \right] dy dz$$

$$= \dot{m}_{\text{ABCD}} - \dot{m}_{\text{EFGH}}$$

$$= \left[\int \rho dy dz \times u \right] - \left[\int \rho dy dz - u \right] - dx \left[\frac{\partial(\rho u)}{\partial z} dy dz \right]$$

$$= - dx \left[\frac{\partial(\rho u)}{\partial z} dy dz \right] \quad \text{--- (2)}$$

iii) along y-direction,

$$= - dx \left[\frac{\partial(\rho u)}{\partial y} dx dz \right] \quad \text{--- (3)} \quad \therefore$$

$$= - dz \left[\frac{\partial(\rho u)}{\partial y} dx dz \right] \quad \text{--- (3)}$$

The mass per time of fluid is = Total mass of the fluid Element

$$\therefore m = [\rho x \cdot dy \cdot dz]$$

$$\dot{m} = \frac{\partial}{\partial t} [dx \cdot dy \cdot dz]$$

$$\therefore \frac{\partial}{\partial t} [\rho dx dy dz] = dx dy dz \left[\frac{\partial(\rho)}{\partial x} + \frac{\partial(\rho)}{\partial y} + \frac{\partial(\rho)}{\partial z} \right]$$

$$\boxed{\frac{\partial \rho}{\partial t} + \frac{\partial(\rho u)}{\partial x} + \frac{\partial(\rho v)}{\partial y} + \frac{\partial(\rho w)}{\partial z} = 0}$$

The above Equation, is called Continuity Equation in 3-D.

for compressible li.

for Non, Compressible liquid.

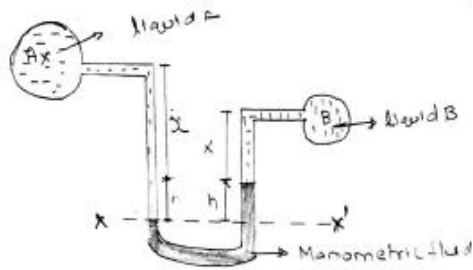
Density remain constant

$$\therefore \frac{\partial u}{\partial x} + \frac{\partial v}{\partial y} + \frac{\partial w}{\partial z} = 0$$

5 Marks

2 Marks

4 With a neat diagram, explain U-tube differential manometers and inverted U-tube differential manometers [10]



2 Marks

i) U-tube differential Manometer.

Differential Manometer is used to find the Pressure at different points.

Let, $x-x'$ be the reference line

$x \rightarrow$ be the distance of liquid A from reference line

$y \rightarrow$ be the distance of liquid B from reference line.

$P_A \rightarrow$ Pressure of liquid A with density ρ_A

$P_B \rightarrow$ Pressure of liquid B with density ρ_B

$h \rightarrow$ height of Manometric fluid from reference line

$\rho \rightarrow$ density of manometric fluid.

The manometric fluid which is used should have density higher than the other two liquids.

Applying Pascal's Law to Right limb

$$P_B + \rho_B y + \rho g h = 0 \quad \text{--- (1)}$$

Applying Pascal's law to left limb.

$$P_A + \rho_A x = 0 \quad \text{--- (2)}$$

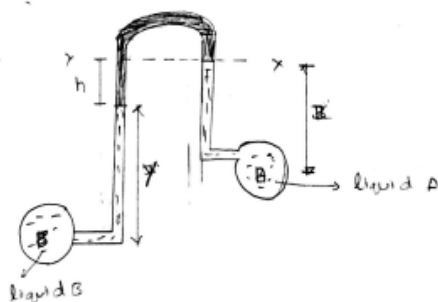
$$\text{eq (2) - eq (1)}$$

$$P_A - P_B = -\rho_B y - \rho g h + \rho_A x$$

$$\boxed{P_A - P_B = \rho_A x - \rho_B y - \rho g h}$$

3 Marks

b) Inverted U-tube differential Manometer.



2 Marks

Let,

x - be the distance of liquid A from 'xx'

y - be the distance of liquid B from 'xx'

h - be the distance of Manometric
fluid from 'xx'

P_A & P_B → Pressure of liquid A and B with
density ρ_A and ρ_B

Applying Pascal's law to left limb

$$P_B - \int_2 \rho_B g y - \int_2 \rho_B g h = 0 \quad \text{--- (1)}$$

Applying Pascal's law to Right limb

$$P_A - \rho_A g x = 0 \quad \text{--- (2)}$$

Eq (2) - (1)

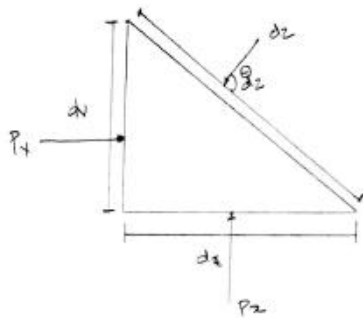
$$P_A - P_B = \int_2 \rho_B g y + \int_2 \rho_B g h - \rho_A g x$$

3 Marks

5 State and prove Pascal law and also show that the Pressure at any point in fluid depends on height of the fluid [10]

Pascal's law

→ The pressure of the liquid at any point will
be same from all directions



Consider a wedge shaped fluid element of dimensions dx , dy and z let, P_x , P_y and P_z be the
Pressure applied in x , y and z direction.

Resolving along x - axis

$$P_x dx - P_z dz \sin(90^\circ) = 0$$

$$P_x dx - P_z dz = 0$$

$$dx (P_x - P_z) = 0$$

$$P_x - P_z = 0 \quad \text{--- (1)}$$

2 Marks

Resolving in y -direction

$$P_y dy (1) - P_2 ds \cos(90-\theta) = 0$$

$$P_y dy - P_2 dy = 0$$

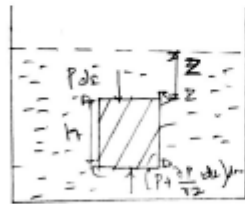
$$dy [P_y - P_2] = 0$$

$$P_y - P_2 = 0 \quad \text{--- (2)}$$

from equation (1) and (2)

$$P_y + P_z + P_x = 0$$

$$P_y = P_z = P_x$$



Consider a liquid element placed in a block at a distance z from the top surface.

The pressure exerted on surface AB = $P dA$

The pressure exerted on surface CD = $(P + \frac{\partial P}{\partial z} dz) \cdot dA$

$$\therefore P dA - (P + \frac{\partial P}{\partial z} dz) + \rho g h \cdot dz$$

$$-\frac{\partial P}{\partial z} dz = -\rho g dz$$

$$\therefore -\partial p = -\rho g \partial z$$

$$= \partial p = \rho g \partial z$$

$$\therefore \boxed{p = \rho g z}$$

The above equation tells ρg is constant value and pressure is depend on ' z ' (i.e. height of fluid).

3 Marks

2 Marks

3 Marks

PART - C (Answer all questions)

- 6 A U-tube differential manometer containing mercury is connected on one side to pipe A containing carbon tetrachloride (Sp.Gr 1.6) under a pressure of 120kPa, and on the other side to pipe B containing oil (Sp.Gr 0.8) under a pressure of 200kPa. The pipe A lies 2.5m above pipe B and the mercury level in the limb communicating with pipe A lies 4m below the pipe A. Determine the difference in the levels of mercury in the two limbs of the manometer.

[10]

$$P_A = 120 \text{ kPa} = 120 \times 10^3 \text{ Pa}$$

$$P_B = 200 \text{ kPa} = 200 \times 10^3 \text{ Pa}$$

$$\rho_A = \delta \rho \cdot G_1 \times \rho_{\text{std}}$$

$$\rho_A = 1.6 \times 1000$$

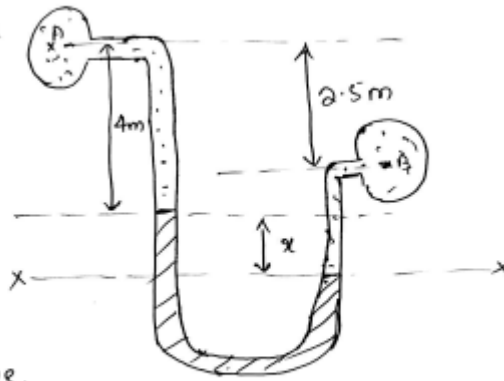
$$= 1600 \text{ kg/m}^3$$

$$\rho_B = 0.8 \times \rho_{\text{std}}$$

$$= 800 \text{ kg/m}^3$$

$$\rho = 13600 \text{ kg/m}^3$$

From figure we have,



$$P_A + (\rho_A \times g \times 4) + \rho g x = P_B + \rho_B g ((4 - 2.5) + x)$$

$$120 \times 10^3 + 4(1600 \times 9.8 \times 4) + (13600 \times 9.8 \times x)$$

$$= 200 \times 10^3 + [800 \times 9.8 (1.5 + x)]$$

$$120000 + 133280x - 7840x - 11760$$

$$= 200000 - 120000 - 62720$$

$$\Rightarrow 125440x = 29040$$

$$\Rightarrow x = \underline{\underline{0.2315 \text{ m}}}$$

Data- 2
Marks

Diagram- 2
Marks

Solution- 6
Marks

- 7 Is the motion $u=x^2y$; $v=2yz-xy^2$; $w=x^2-z^2$ kinematically possible for steady flow of an incompressible fluid. [10]

$$u = x^2y$$

$$v = 2yz - xy^2$$

$$w = x^2 - z^2$$

for Compressible fluid we need to check

$$\frac{\partial u}{\partial x} + \frac{\partial v}{\partial y} + \frac{\partial w}{\partial z} = 0 \quad (1)$$

$$\therefore u = x^2y \Rightarrow$$

Partially diff w.r.t x.

$$\frac{\partial u}{\partial x} = 2xy \quad (2)$$

$$v = 2yz - xy^2$$

Partially diff w.r.t y

$$\frac{\partial v}{\partial y} = 2z - 2xy \quad (3)$$

$$\text{Partially } w = x^2 - z^2$$

Partially diff w.r.t z

$$\frac{\partial w}{\partial z} = -2z \quad (4)$$

Substituting eq (2), (3), (4) in Eq (1)

$$\therefore 2xy + [2z - 2xy] + [-2z]$$

$$2x/y + 2z - 2x/y - 2z$$

$$= 0$$

\therefore Hence the Continuity Equation is Satisfied for

laminar

Hence the given motion, is possible for

Steady flow for incompressible fluid

OR

Identifying the continuity equation- 2 Marks

Differentiation - 5 Marks

Substitution and Proof- 3 Marks

Given $u = \frac{kx}{(x^2+y^2+z^2)^{3/2}}$ and $v = \frac{ky}{(x^2+y^2+z^2)^{3/2}}$, find an expression for velocity component w in a three-dimensional incompressible fluid flow. [10]

$$u = \frac{kx}{(x^2+y^2+z^2)^{3/2}} \quad v = \frac{ky}{(x^2+y^2+z^2)^{3/2}} \quad w = ?$$

$$\frac{\partial u}{\partial x} = k \left[\frac{(x^2+y^2+z^2)^{3/2} (1) - x \left(\frac{3}{2}\right) (x^2+y^2+z^2)^{1/2} \cdot 2x}{(x^2+y^2+z^2)^3} \right]$$

$$= \frac{(x^2+y^2+z^2)^{1/2} k \left[x^2+y^2+z^2 - \frac{3x^2}{2} \right]}{(x^2+y^2+z^2)^3}$$

$$= \frac{k \left[(x^2+y^2+z^2) - 3x^2 \right]}{(x^2+y^2+z^2)^{5/2}} \quad \rightarrow$$

$$\frac{\partial v}{\partial y} = k \left[\frac{(x^2+y^2+z^2)^{3/2} (1) - y \left(\frac{3}{2}\right) (x^2+y^2+z^2)^{1/2} \cdot 2y}{(x^2+y^2+z^2)^3} \right]$$

$$= \frac{k \left[x^2+y^2+z^2 - 3y^2 \right]}{(x^2+y^2+z^2)^{5/2}}$$

We have $\frac{\partial u}{\partial x} + \frac{\partial v}{\partial y} + \frac{\partial w}{\partial z} = 0$ for an incompressible fluid

$$\therefore \frac{\partial w}{\partial z} = -k \left[\frac{x^2+y^2+z^2 - 3x^2}{(x^2+y^2+z^2)^{5/2}} + \frac{x^2+y^2+z^2 - 3y^2}{(x^2+y^2+z^2)^{5/2}} \right]$$

$$= -k \left[\frac{-x^2 - y^2 + 2z^2}{(x^2+y^2+z^2)^{5/2}} \right]$$

$$= \frac{k \left(x^2 + y^2 - 2z^2 \right)}{(x^2+y^2+z^2)^{5/2}}$$

$$\frac{\partial w}{\partial z} = \frac{k \left(x^2 + y^2 + z^2 - 3z^2 \right)}{(x^2+y^2+z^2)^{5/2}} \quad \text{--- (1)}$$

Integrating the eqn (1) we get

$$w = \frac{kz}{(x^2+y^2+z^2)^{3/2}}$$

Differentiation - 4 Marks

Substitution and identification - 3 Marks

Integration - 3 Marks