

**Internal Assessment Test II – April 2018**

<b>Sub:</b> Finite Element Methods	Max	
Date: <u>16/04/2018</u> Duration: <u>90 mins</u>	Marks: <u>50</u>	Sem: <u>VI</u>

<b>Code:</b> 15ME61
<b>Branch:</b> MECH

**Marks**      OBE  
CO      RBT

**1 a** The nodal co-ordinates of a triangular element are shown in fig 1. The x - coordinates of an interior point P is 3.3 and shape function.  $N_1 = 0.3$ . Determine  $N_2$ ,  $N_3$  and y - coordinate point P.

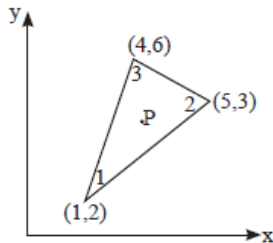


fig 1

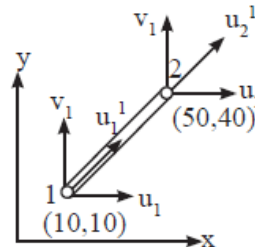


fig 2

**04**      CO3      L2

**b** For the truss element shown in below fig 2 (x, y) co-ordinates of the element are indicated near nodes 1, 2. The element displacement dof vector is given by  $[u] = [1.5, 1.0, 2.1, 4.3]^T \times 10^{-2} \text{ mm}$  Take  $E = 300 \times 10^3 \text{ N/mm}^2$   
 $A = 100 \text{ mm}^2$  Determine the following

**06**      CO4      L2

- Element displacement dof in local co-ordinates
- Stress in the element
- Stiffness matrix of the element

**2** Derive Shape function and Jacobian matrix for CST element

**10**      CO3      L3

**3** Derive shape functions for QUAD 4 element using Lagrangian element

**10**      CO3      L3

**4** Derive stiffness matrix for 2D truss

**10**      CO4      L3

**5** For the two-bar truss shown in fig 3. Determine the nodal displacements element stresses and support reactions. A force of  $P = 1000 \text{ KN}$  is applied at node 1. Assume  $E = 210 \text{ GPa}$  and  $A = 600 \text{ mm}^2$  for each element.

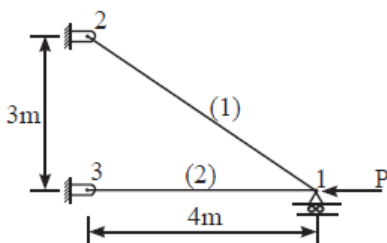


fig 3

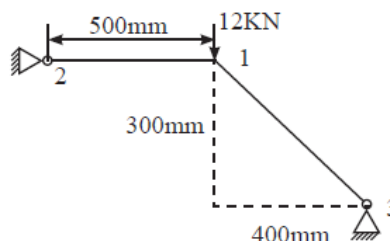


fig 4

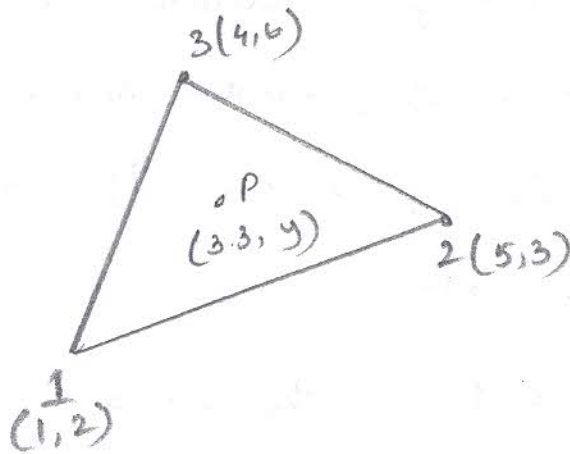
**10**      CO4      L4

**6** Determine the unknown displacements and stresses for the structure shown in fig 4. Take  $E = 0.7 \times 10^5 \text{ MPa}$ ,  $A = 200 \text{ mm}^2$ .

**10**      CO4      L4

**Solution of Internal Assessment Test II**

1 a



$$x = N_1 x_1 + N_2 x_2 + N_3 x_3$$

$$3.3 = 0.3(1) + \eta(5) + (1 - \xi - \eta)4$$

$$3.3 = 0.3 + 5\eta + 4 - 4\xi - 4\eta$$

$$-4\xi + \eta = -1$$

$$-4(0.3) + \eta = -1$$

$$\eta = 0.2$$

$$\therefore N_2 = \eta = 0.2 //$$

$$N_3 = 1 - \xi - \eta = 1 - 0.3 - 0.2 = 0.5$$

$$N_3 = 0.5 //$$

Y coordinate

$$y = N_1 y_1 + N_2 y_2 + N_3 y_3$$

$$= 0.3(2) + 0.2(3) + 0.5(6)$$

$$y = 4.2 //$$

1 b

## Node Data

Node No	$x$ (mm)	$y$ (mm)
1	10	10
2	50	40

## Element Connectivity Table

Element No	Initial Node	Final Node	Length of element	$l$	$m$
1	1	2	50	0.8	0.6

$$l_e = \sqrt{(x_2 - x_1)^2 + (y_2 - y_1)^2} = \sqrt{(50 - 10)^2 + (40 - 10)^2}$$

$$l_e = 50 \text{ mm}$$

$$l = \frac{x_{FN} - x_{IN}}{L} = \frac{40}{50} = 0.8$$

$$m = \frac{y_{FN} - y_{IN}}{L} = \frac{30}{50} = 0.6$$

i) Element displacement dof in local coordinate

$$a' = L q$$

$$\begin{bmatrix} a'_1 \\ a'_2 \end{bmatrix} = \begin{bmatrix} l & m & 0 & 0 \\ 0 & 0 & l & m \end{bmatrix} \begin{bmatrix} q_1 \\ q_2 \\ q_3 \\ q_4 \end{bmatrix}$$

$$= \begin{bmatrix} 0.8 & 0.6 & 0 & 0 \\ 0 & 0 & 0.8 & 0.6 \end{bmatrix} \begin{bmatrix} 1.5 \\ 1 \\ 2.1 \\ 4.3 \end{bmatrix} \times 10^{-2}$$

$$\begin{bmatrix} a'_1 \\ a'_2 \end{bmatrix} = \begin{bmatrix} 0.018 \\ 0.0426 \end{bmatrix} \text{ mm} //$$

ii) Stress  $\sigma = E \cdot \frac{1}{L} [-l \quad -m \quad l \quad m] \begin{bmatrix} q_1 \\ q_2 \\ q_3 \\ q_4 \end{bmatrix}$

$$= 300 \times 10^3 \times \frac{1}{50} [-0.8 \quad -0.6 \quad 0.8 \quad 0.6] \begin{bmatrix} 1.5 \\ 1 \\ 2.1 \\ 4.3 \end{bmatrix} \times 10^{-2}$$

$$\sigma = 147.6 \text{ N/mm}^2 //$$

## Stiffness matrix

$$K = \frac{EA}{le} \begin{bmatrix} l^2 & lm & -l^2 & -lm \\ lm & m^2 & -lm & -m^2 \\ -l^2 & -lm & l^2 & lm \\ -lm & -m^2 & lm & m^2 \end{bmatrix}$$

$$= \frac{300 \times 10^3 \times 100}{50} \begin{bmatrix} 0.64 & 0.48 & -0.64 & -0.48 \\ 0.48 & 0.36 & -0.48 & -0.36 \\ -0.64 & -0.48 & 0.64 & 0.48 \\ -0.48 & -0.36 & 0.48 & 0.36 \end{bmatrix}$$

$$K = 10^5 \begin{bmatrix} 3.84 & 2.88 & -3.84 & -2.88 \\ 2.88 & 2.16 & -2.88 & -2.16 \\ -3.84 & -2.88 & 3.84 & 2.88 \\ -2.88 & -2.16 & 2.88 & 2.16 \end{bmatrix} //$$

2

Let the shape function  $N_1$  be

$$N_1 = \alpha_1 + \alpha_2 \xi + \alpha_3 \eta$$

At node 1,  $N_1 = 1$ ,  $\xi = 1$ ,  $\eta = 0$

$$1 = \alpha_1 + \alpha_2 \rightarrow \textcircled{1}$$

At node 2,  $N_1 = 0$ ,  $\xi = 0$ ,  $\eta = 1$

$$0 = \alpha_1 + \alpha_3 \rightarrow \textcircled{2}$$

At node 3,  $N_1 = 0$ ,  $\xi = 0$ ,  $\eta = 0$

$$\alpha_1 = 0 \rightarrow \textcircled{3}$$

$$\alpha_3 = 0 ; \alpha_2 = 1$$

$$\boxed{N_1 = \xi}$$

Let the shape function  $N_2$  be

$$N_2 = \alpha_1 + \alpha_2 \xi + \alpha_3 \eta$$

At node 1,  $N_2 = 0$ ,  $\xi = 1$ ,  $\eta = 0$

$$0 = \alpha_1 + \alpha_2 \rightarrow \textcircled{1}$$

At node 2,  $N_2 = 1$ ,  $\xi = 0$ ,  $\eta = 1$

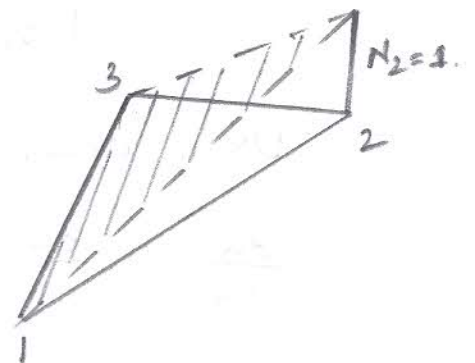
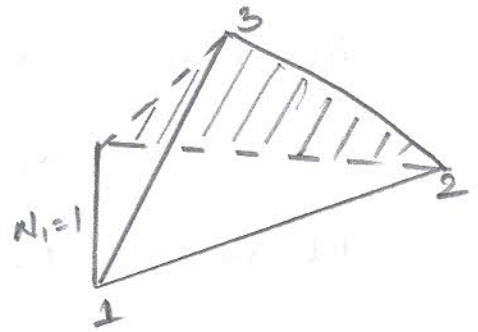
$$1 = \alpha_1 + \alpha_3 \rightarrow \textcircled{2}$$

At node 3,  $N_2 = 0$ ,  $\xi = 0$ ,  $\eta = 0$

$$\alpha_1 = 0 \rightarrow \textcircled{3}$$

$$\alpha_2 = 0, \alpha_3 = 1.$$

$$\boxed{N_2 = \eta}$$



Let the shape function  $N_3$  be

$$N_3 = \alpha_1 + \alpha_2 \xi + \alpha_3 \eta$$

At node 1,  $N_3 = 0$ ,  $\xi = 1$ ,  $\eta = 0$

$$0 = \alpha_1 + \alpha_2 \rightarrow \textcircled{1}$$

At node 2,  $N_3 = 0$ ,  $\xi = 0$ ,  $\eta = 1$

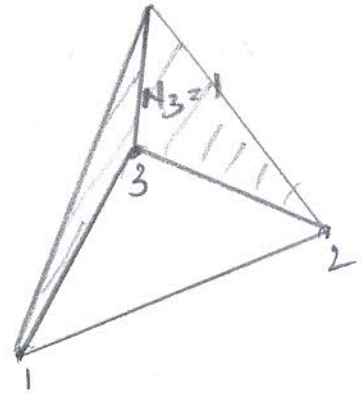
$$0 = \alpha_1 + \alpha_3 \rightarrow \textcircled{2}$$

At node 3,  $N_3 = 1$ ,  $\xi = 0$ ,  $\eta = 0$

$$\alpha_1 = 1 \rightarrow \textcircled{3}$$

$$\alpha_2 = -1, \quad \alpha_3 = -1$$

$$\boxed{N_3 = 1 - \xi - \eta}$$



### DERIVATION OF STRAIN MATRIX FOR CST

Strain can be expressed as

$$\epsilon = \begin{bmatrix} \epsilon_x \\ \epsilon_y \\ \gamma_{xy} \end{bmatrix} = \begin{bmatrix} \partial u / \partial x \\ \partial v / \partial y \\ \frac{\partial u}{\partial y} + \frac{\partial v}{\partial x} \end{bmatrix} \rightarrow \textcircled{1}$$

Using chain rule of partial differentiation

$$\frac{\partial u}{\partial \xi} = \frac{\partial u}{\partial x} \cdot \frac{\partial x}{\partial \xi} + \frac{\partial u}{\partial y} \cdot \frac{\partial y}{\partial \xi}$$

$$\frac{\partial u}{\partial \eta} = \frac{\partial u}{\partial x} \cdot \frac{\partial x}{\partial \eta} + \frac{\partial u}{\partial y} \cdot \frac{\partial y}{\partial \eta}$$

$$\begin{bmatrix} \frac{\partial u}{\partial \xi} \\ \frac{\partial u}{\partial \eta} \end{bmatrix} = \begin{bmatrix} \frac{\partial x}{\partial \xi} & \frac{\partial y}{\partial \xi} \\ \frac{\partial x}{\partial \eta} & \frac{\partial y}{\partial \eta} \end{bmatrix} \begin{bmatrix} \frac{\partial u}{\partial x} \\ \frac{\partial u}{\partial y} \end{bmatrix}$$

$$\begin{bmatrix} \frac{\partial u}{\partial \xi} \\ \frac{\partial u}{\partial \eta} \end{bmatrix} = \mathcal{J} \begin{bmatrix} \frac{\partial u}{\partial x} \\ \frac{\partial u}{\partial y} \end{bmatrix} \rightarrow \textcircled{2} \quad \text{Where } \mathcal{J} \rightarrow \text{Jacobian}$$

$$\mathcal{J} = \begin{bmatrix} \frac{\partial x}{\partial \xi} & \frac{\partial y}{\partial \xi} \\ \frac{\partial x}{\partial \eta} & \frac{\partial y}{\partial \eta} \end{bmatrix}$$

Jacobian is a scaling matrix which correlates b/w Cartesian coordinate & natural coordinate w.r.t the differential of the displacements.

Using isoparametric formulation

$$x = N_1 x_1 + N_2 x_2 + N_3 x_3$$

$$x = \xi x_1 + \eta x_2 + (1 - \xi - \eta) x_3$$

$$\frac{\partial x}{\partial \xi} = x_1 - x_3 = x_{13}$$

$$\frac{\partial x}{\partial \eta} = x_2 - x_3 = x_{23}$$

$$\text{Similarly } y = N_1 y_1 + N_2 y_2 + N_3 y_3$$

$$y = \xi y_1 + \eta y_2 + (1 - \xi - \eta) y_3$$

$$\frac{\partial y}{\partial \xi} = y_1 - y_3 = y_{13}$$

$$\frac{\partial y}{\partial \eta} = y_2 - y_3 = y_{23}$$



$$N_i = \frac{1}{4} (1 + \xi \xi_i) (1 + \eta \eta_i)$$

$$i = 1, 2, 3, 4$$

$$N_1 = \frac{1}{4} (1 - \xi) (1 - \eta)$$

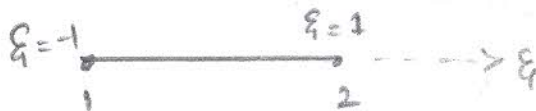
$$N_2 = \frac{1}{4} (1 + \xi) (1 - \eta)$$

$$N_3 = \frac{1}{4} (1 + \xi) (1 + \eta)$$

$$N_4 = \frac{1}{4} (1 - \xi) (1 + \eta)$$

### LAGRANGEAN METHOD

$$N_i(\xi, \eta) = L_i(\xi) L_i(\eta)$$



$$L_1(\xi) = \frac{\xi - \xi_2}{\xi_1 - \xi_2} = \frac{\xi - 1}{-1 - 1}$$

$$L_1(\xi) = \frac{1 - \xi}{2}$$

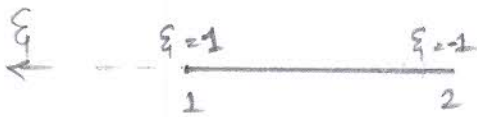


$$L_1(\eta) = \frac{\eta - \eta_4}{\eta_1 - \eta_4} = \frac{\eta - 1}{-1 - 1} = \frac{1 - \eta}{2}$$

$$\begin{aligned} N_1 &= L_1(\xi) L_1(\eta) \\ &= \frac{1 - \xi}{2} \cdot \frac{1 - \eta}{2} \end{aligned}$$

$$N_1 = \frac{1}{4} (1 - \xi)(1 - \eta)$$

$$N_2(\xi, \eta) = L_2(\xi) L_2(\eta)$$



$$L_2(\xi) = \frac{\xi - \xi_1}{\xi_2 - \xi_1} = \frac{\xi - 1}{-1 - 1} = \frac{1 + \xi}{2}$$



$$L_2(\eta) = \frac{\eta - \eta_3}{\eta_2 - \eta_3} = \frac{\eta + 1}{-1 - 1} = \frac{1 - \eta}{2}$$

$$N_2 = \frac{1 + \xi}{2} \cdot \frac{1 - \eta}{2}$$

$$N_2 = \frac{1}{4} (1 + \xi)(1 - \eta)$$

$$N_3(\xi, \eta) = L_3(\xi) L_3(\eta)$$



$$L_3(\xi) = \frac{\xi - \xi_4}{\xi_3 - \xi_4} = \frac{\xi + 1}{+1 + 1} = \frac{1 + \xi}{2}$$



$$L_3(\eta) = \frac{\eta - \eta_2}{\eta_3 - \eta_2} = \frac{\eta + 1}{1 + 1} = \frac{1 + \eta}{2}$$

$$N_3 = \frac{1}{4} (1+\xi) (1+\eta)$$

$$N_4(\xi, \eta) = L_4(\xi) L_4(\eta)$$



$$L_4(\xi) = \frac{\xi - \xi_3}{\xi_{44} - \xi_3} = \frac{\xi - 1}{-1 - 1} = \frac{1 - \xi}{2}$$



$$L_4(\eta) = \frac{\eta - \eta_1}{\eta_4 - \eta_1} = \frac{\eta + 1}{1 + 1} = \frac{1 + \eta}{2}$$

$$N_4 = \frac{1}{4} (1 - \xi) (1 + \eta)$$

## DERIVATION OF ELEMENTAL STIFFNESS MATRIX FOR A TRUSS ELEMENT

Relation b/w nodal displacement of a truss element in local coordinates & global coordinate is expressed as

$$q' = L q$$

where  $q' = \begin{bmatrix} q'_1 \\ q'_2 \end{bmatrix}$ ;  $L = \begin{bmatrix} l & m & 0 & 0 \\ 0 & 0 & l & m \end{bmatrix}$ ;  $q = \begin{bmatrix} q_1 \\ q_2 \\ q_3 \\ q_4 \end{bmatrix}$

A truss element in local coordinate is equivalent to 1D bar element having the stiffness matrix

$$K' = \frac{EA}{l_e} \begin{bmatrix} 1 & -1 \\ -1 & 1 \end{bmatrix}$$

Strain energy for a truss element in local coordinate is given by

$$U_e = \frac{1}{2} q'^T K' q' \rightarrow \textcircled{1}$$

It is required to determine strain energy of truss element in global coordinates.

$$q' = L q$$

Sub. this in eqn  $\textcircled{1}$  we get

$$U_e = \frac{1}{2} q^T L^T K' L q$$

$$U_e = \frac{1}{2} q^T (L^T K' L) q$$

$$U_e = \frac{1}{2} q^T K_e q$$

Where  $K_e$  - elemental stiffness matrix in global coordinates

$$K_e = L^T K' L$$

$$L = \begin{bmatrix} l & m & 0 & 0 \\ 0 & 0 & l & m \end{bmatrix}; \quad L^T = \begin{bmatrix} l & 0 \\ m & 0 \\ 0 & l \\ 0 & m \end{bmatrix}; \quad K' = \frac{EA}{l_e} \begin{bmatrix} 1 & -1 \\ -1 & 1 \end{bmatrix}$$

$$K_e = \begin{bmatrix} l & 0 \\ m & 0 \\ 0 & l \\ 0 & m \end{bmatrix} \frac{EA}{l_e} \begin{bmatrix} 1 & -1 \\ -1 & 1 \end{bmatrix} \begin{bmatrix} l & m & 0 & 0 \\ 0 & 0 & l & m \end{bmatrix}$$

$$K_e = \frac{EA}{l_e} \begin{bmatrix} l^2 & lm & -l^2 & -lm \\ lm & m^2 & -lm & -m^2 \\ -l^2 & -lm & l^2 & lm \\ -lm & -m^2 & lm & m^2 \end{bmatrix}$$

Where  $l$  &  $m$  are direction cosines.

5

Nodal Data

Node No.	x (mm)	y (mm)
1	4000	0
2	0	3000
3	0	0

Element Connectivity Table

Element NO	Initial Node	Final Node	length of element $l_e$ (mm)	$l$	$m$
1	2	1	5000	-0.8	-0.6
2	1	3	4000	-1	0

Element Stiffness matrix

$$K = \frac{EA}{l_e} \begin{bmatrix} l^2 & lm & -l^2 & -lm \\ lm & m^2 & -lm & -m^2 \\ -l^2 & -lm & l^2 & lm \\ -lm & -m^2 & lm & m^2 \end{bmatrix}$$

For element 1

$$K_1 = \frac{210 \times 10^3 \times 600}{5000}$$

$$\begin{bmatrix} 0.64 & -0.48 & -0.64 & 0.48 \\ -0.48 & 0.36 & 0.48 & -0.36 \\ -0.64 & 0.48 & 0.64 & -0.48 \\ 0.48 & -0.36 & -0.48 & 0.36 \end{bmatrix}$$

$$K_1 = 10^3 \begin{bmatrix} 1 & 2 & 3 & 4 \\ 16.128 & -12.096 & -16.128 & 12.096 \\ -12.096 & 9.072 & 12.096 & -9.072 \\ -16.128 & 12.096 & 16.128 & -12.096 \\ 12.096 & -9.072 & -12.096 & 9.072 \end{bmatrix}$$

For element 2

$$K_2 = \frac{210 \times 10^3 \times 600}{4000} \begin{bmatrix} 1 & 0 & -1 & 0 \\ 0 & 0 & 0 & 0 \\ -1 & 0 & 1 & 0 \\ 0 & 0 & 0 & 0 \end{bmatrix}$$

$$= 10^3 \begin{bmatrix} 1 & 2 & 5 & 6 \\ 31.5 & 0 & -31.5 & 0 \\ 0 & 0 & 0 & 0 \\ -31.5 & 0 & 31.5 & 0 \\ 0 & 0 & 0 & 0 \end{bmatrix}$$

Global stiffness matrix

$$K = 10^3 \begin{bmatrix} 1 & 2 & 3 & 4 & 5 & 6 \\ 47.628 & -12.096 & -16.128 & 12.096 & -31.5 & 0 \\ -12.096 & 9.072 & 12.096 & -9.072 & 0 & 0 \\ -16.128 & 12.096 & 16.128 & -12.096 & 0 & 0 \\ 12.096 & -9.072 & -12.096 & 9.072 & 0 & 0 \\ -31.5 & 0 & 0 & 0 & 31.5 & 0 \\ 0 & 0 & 0 & 0 & 0 & 0 \end{bmatrix}$$

Equilibrium eqn.

$$[K][q] = [F]$$

$$10^3 \begin{bmatrix} 47.628 & -12.096 & -16.128 & 12.096 & -31.5 & 0 \\ 12.096 & 9.072 & 12.096 & -9.072 & 0 & 0 \\ -16.128 & 12.096 & 16.128 & -12.096 & 0 & 0 \\ 12.096 & -9.072 & -12.096 & 9.072 & 0 & 0 \\ -31.5 & 0 & 0 & 0 & 31.5 & 0 \\ 0 & 0 & 0 & 0 & 0 & 0 \end{bmatrix} \begin{bmatrix} q_1 \\ q_2 \\ q_3 \\ q_4 \\ q_5 \\ q_6 \end{bmatrix} = \begin{bmatrix} -106 \\ 0 \\ 0 \\ 0 \\ 0 \\ 0 \end{bmatrix}$$

Nodal displacement

$$q_1 = -20.996 \text{ mm//}$$

$$q_2 = q_3 = q_4 = q_5 = q_6 = 0//$$

Stress

$$\sigma = \frac{E}{le} \begin{bmatrix} -l & -m & l & m \end{bmatrix} \begin{bmatrix} q_1 \\ q_2 \\ q_3 \\ q_4 \end{bmatrix}$$

$$= \frac{210 \times 10^3}{5000} \begin{bmatrix} 0.8 & -0.6 & -0.8 & 0.6 \end{bmatrix} \begin{bmatrix} -20.996 \\ 0 \\ 0 \\ 0 \end{bmatrix}$$

$$\sigma_1 = -705.46 \text{ N/mm}^2//$$



$$\sigma_2 = \frac{210 \times 10^3}{4000} \begin{bmatrix} 1 & 0 & -1 & 0 \end{bmatrix} \begin{bmatrix} -20.996 \\ 0 \\ 0 \\ 0 \end{bmatrix}$$

$$= -1102.96 \text{ N/mm}^2 //$$

Reaction

$$R_2 = K_{21} q_1 = -12.096 \times 10^3 \times -20.996 = 253.96 \text{ kN} //$$

$$R_3 = K_{31} q_1 = -16.126 \times 10^3 \times -20.996 = 338.62 \text{ kN} //$$

$$R_4 = K_{41} q_1 = -253.96 \text{ kN} //$$

$$R_5 = K_{51} q_1 = -31.5 \times 10^3 \times -20.996 = 661.37 \text{ kN} //$$

6

Node Data		
Node No.	x(mm)	y(mm)
1	500	300
2	0	300
3	900	300

Element Connectivity Table

Element No	Initial Node	Final Node	length of element 'le'	l	m
1	1	2	500	-1	0
2	1	3	500	0.8	-0.6

$$l = \frac{x_{F.N} - x_{I.N}}{le} = \frac{0 - 500}{500} = -1$$

$$m = \frac{y_{F.N} - y_{I.N}}{le} = \frac{300 - 300}{500} = 0$$

Stiffness matrix

$$K = \frac{EA}{le} \begin{bmatrix} l^2 & lm & -l^2 & -lm \\ lm & m^2 & -lm & -m^2 \\ -l^2 & -lm & l^2 & lm \\ -lm & -m^2 & lm & m^2 \end{bmatrix}$$

For element 1

$$K_1 = \frac{0.7 \times 10^5 \times 200}{500} \begin{bmatrix} 1 & 0 & -1 & 0 \\ 0 & 0 & 0 & 0 \\ -1 & 0 & 1 & 0 \\ 0 & 0 & 0 & 0 \end{bmatrix}$$

$$K_1 = 10^3 \begin{bmatrix} 28 & 0 & -28 & 0 \\ 0 & 0 & 0 & 0 \\ -28 & 0 & 28 & 0 \\ 0 & 0 & 0 & 0 \end{bmatrix}$$

For element 2

$$K_2 = \frac{0.7 \times 10^5 \times 200}{500} \begin{bmatrix} 0.64 & -0.48 & -0.64 & 0.48 \\ -0.48 & 0.36 & 0.48 & -0.36 \\ -0.64 & 0.48 & 0.64 & -0.48 \\ 0.48 & -0.36 & -0.48 & 0.36 \end{bmatrix}$$

$$= 10^3 \begin{bmatrix} 17.92 & -13.44 & -17.92 & 13.44 \\ -13.44 & 10.08 & 13.44 & -10.08 \\ -17.92 & 13.44 & 17.92 & -13.44 \\ 13.44 & -10.08 & -13.44 & 10.08 \end{bmatrix}$$

Global

$$K = 10^3 \begin{bmatrix} 45.92 & -13.44 & -28 & 0 & 0 & 0 \\ -13.44 & 10.08 & 0 & 0 & 13.44 & -10.08 \\ -28 & 0 & 28 & 0 & 0 & 0 \\ 0 & 0 & 0 & 0 & 0 & 0 \\ 0 & 13.44 & 0 & 0 & 17.92 & -13.44 \\ 0 & -10.08 & 0 & 0 & -13.44 & 10.08 \end{bmatrix}$$

$$[K][q] = [F]$$

$$10^3 \begin{bmatrix} 45.92 & -13.44 & -28 & 0 & 0 & 0 \\ -13.44 & 10.08 & 0 & 0 & 13.44 & -10.08 \\ -28 & 0 & 28 & 0 & 0 & 0 \\ 0 & 0 & 0 & 0 & 0 & 0 \\ 0 & 13.44 & 0 & 0 & 17.92 & -13.44 \\ 0 & -10.08 & 0 & 0 & -13.44 & 10.08 \end{bmatrix} \begin{bmatrix} q_1 \\ q_2 \\ q_3 \\ q_4 \\ q_5 \\ q_6 \end{bmatrix} = \begin{bmatrix} 0 \\ -12 \times 10^3 \\ 0 \\ 0 \\ 0 \\ 0 \end{bmatrix}$$

$$q_3 = q_4 = q_5 = q_6 = 0$$

$$q_1 = -0.57 \text{ mm} //, \quad q_2 = -1.95 \text{ mm} //$$

Stresses:

$$\sigma_1 = E \cdot \frac{1}{le} \begin{bmatrix} -l & -m & l & m \end{bmatrix} \begin{bmatrix} q_1 \\ q_2 \\ q_3 \\ q_4 \end{bmatrix}$$

$$= 0.7 \times 10^5 \times \frac{1}{500} \begin{bmatrix} 1 & 0 & -1 & 0 \end{bmatrix} \begin{bmatrix} -0.57 \\ -1.95 \\ 0 \\ 0 \end{bmatrix}$$

$$= -79.8 \text{ N/mm}^2$$

$$\sigma_2 = E \cdot \frac{1}{le} \begin{bmatrix} -l & -m & l & m \end{bmatrix} \begin{bmatrix} q_1 \\ q_2 \\ q_5 \\ q_6 \end{bmatrix}$$

$$= 0.7 \times 10^5 \times \frac{1}{500} \begin{bmatrix} -0.8 & 0.6 & 0.8 & -0.6 \end{bmatrix} \begin{bmatrix} -0.57 \\ -1.95 \\ 0 \\ 0 \end{bmatrix}$$

$$\sigma_2 = -99.96 \text{ N/mm}^2 //$$

Displacements

$$q_1 = -0.57 \text{ mm}$$

$$q_2 = -1.95 \text{ mm}$$

$$q_3 = 0$$

$$q_4 = 0$$

$$q_5 = 0$$

$$q_6 = 0$$

Stress

$$\sigma_1 = -79.8 \text{ N/mm}^2 //$$

$$\sigma_2 = -99.96 \text{ N/mm}^2 //$$