

**Internal Assessment Test II – April 2019**

<b>Sub:</b> Finite Element Methods	<b>Max Marks:</b> 50	<b>Sem:</b> VI
<b>Date:</b> 15/04/2019	<b>Duration:</b> 90 mins	
<b>Note:</b> Answer any five questions.		

<b>Code:</b> 15ME61
<b>Branch:</b> MECH

**Marks** OBE  
CO RBT

- 1 Figure 1 shows a thin plate of uniform thickness of 1mm, young's modulus  $E = 200\text{GPa}$ , weight density of the plate  $76.6 \times 10^{-6} \text{ N/mm}^3$ . In addition to its weight, it is subjected to a point load of 1000N at its mid-point. Model the plate with 2 bar element.

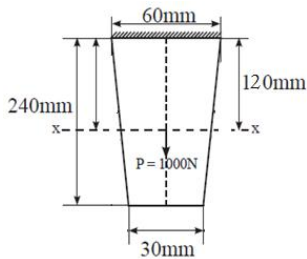


Figure 1

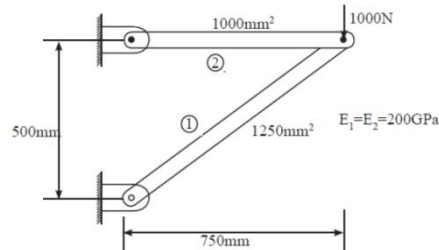


Figure 2

- 2 Derive shape function of Quadratic Bar element in natural co-ordinates. 10 CO3 L3
- 3 Derive the stiffness matrix for truss element in terms of direction cosines. 10 CO4 L3
- 4 For the Pin Jointed Configuration shown in below Figure 2. Formulate the stiffness matrix also determine the nodal displacements. 10 CO4 L4
- 5 Derive shape functions for QUAD 4 element using Lagrangian element. 10 CO3 L3
- 6 a The nodal co-ordinates of a triangular element are shown in Figure 3. The x - coordinates of an interior point P is 3.3 and shape function.  $N_1 = 0.3$ . Determine  $N_2$ ,  $N_3$  and y - coordinate point P. 04 CO3 L2

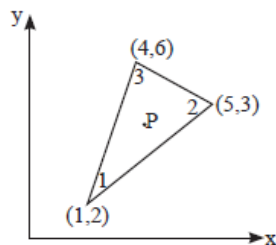


Figure 3

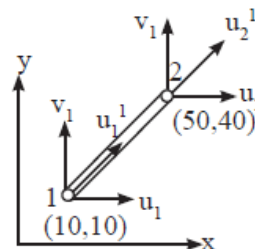


Figure 4

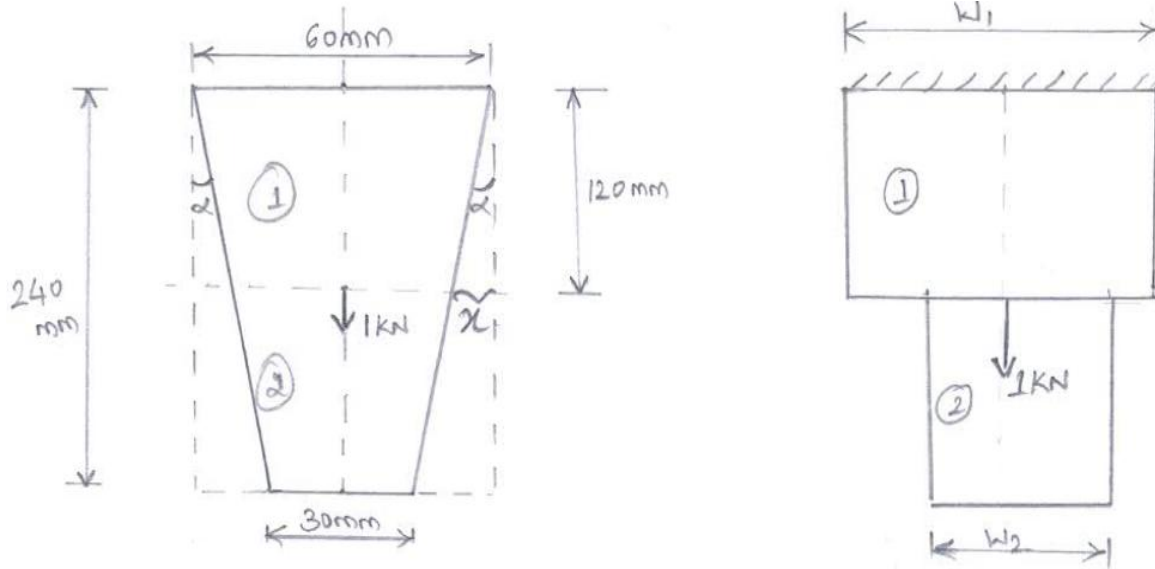
- b For the truss element shown in below Figure 4 (x, y) co-ordinates of the element are indicated near nodes 1, 2. The element displacement dof vector is given by  $[u] = [1.5, 1.0, 2.1, 4.3]^T \times 10^{-2} \text{ mm}$  Take  $E = 300 \times 10^3 \text{ N/mm}^2$   
 $A = 100 \text{ mm}^2$  Determine the following 06 CO4 L2
- Element displacement dof in local co-ordinates
  - Stress in the element
  - Stiffness matrix of the element

IAT 2 - Solution April 2019

Subject : FEM(15ME61)

1. Given:

Thin plate of uniform thickness of 1mm, young's modulus  $E = 200\text{GPa}$ , weight density of the plate  $76.6 \times 10^{-6} \text{ N/mm}^3$



$$E = 200 \text{ GPa} = 200 \times 10^3 \text{ MPa}$$

From fig.

$$\tan \alpha = \frac{15}{240} \quad ; \quad \tan \alpha = \frac{x}{120}$$

$$\alpha = 3.57^\circ$$

$$x = 7.5 \text{ mm}$$

Volume of <sup>tapered</sup> plate ① = Volume of stepped plate ②

$$\text{Volume} = \frac{b}{2} [\text{Addition of two } \parallel \text{ sides}] \times \text{thickness}$$

$$\frac{120}{2} [60 + 45] \times 1 = W_1 \times 120 \times 1$$

$$W_1 = 52.5 \text{ mm} //$$

$$\text{Area} = A_1 = W_1 \times t = 52.5 \times 1$$

$$A_1 = 52.5 \text{ mm}^2 //$$

$$\frac{120}{2} [45 + 30] \times 1 = W_2 \times 120 \times 1$$

$$W_2 = 37.5 \text{ mm} //$$

$$A_2 = W_2 \times t = 37.5 \text{ mm}^2 //$$

Elemental Stiffness matrix

$$K = \frac{EA}{le} \begin{bmatrix} 1 & -1 \\ -1 & 1 \end{bmatrix}$$

For element 1

$$K_1 = \frac{200 \times 10^3 \times 52.5}{120} \begin{bmatrix} 1 & -1 \\ -1 & 1 \end{bmatrix}$$

$$= 10^5 \begin{bmatrix} 0.875 & -0.875 \\ -0.875 & 0.875 \end{bmatrix} \begin{matrix} 1 \\ 2 \end{matrix}$$

For element 2

$$K_2 = \frac{200 \times 10^3 \times 37.5}{120} \begin{bmatrix} 1 & -1 \\ -1 & 1 \end{bmatrix}$$

$$= 10^5 \begin{bmatrix} 0.625 & -0.625 \\ -0.625 & 0.625 \end{bmatrix} \begin{matrix} 2 \\ 3 \end{matrix}$$

Global stiffness matrix

$$K = 10^5 \begin{bmatrix} 0.875 & -0.875 & 0 \\ -0.875 & 1.5 & -0.625 \\ 0 & -0.625 & 0.625 \end{bmatrix} \begin{matrix} 1 \\ 2 \\ 3 \end{matrix}$$

Elemental body forces

$$F = \frac{SA \rho c}{2} \begin{bmatrix} 1 \\ 1 \end{bmatrix}$$

For element 1

$$F_1 = \frac{76.6 \times 10^{-6} \times 52.5 \times 120}{2} \begin{bmatrix} 1 \\ 1 \end{bmatrix}$$

$$= \begin{bmatrix} 0.24129 \\ 0.24129 \end{bmatrix} \begin{matrix} 1 \\ 2 \end{matrix}$$

For element 2

$$F_2 = \frac{76.6 \times 10^{-6} \times 37.5 \times 120}{2} \begin{bmatrix} 1 \\ 1 \end{bmatrix}$$

$$= \begin{bmatrix} 0.17235 \\ 0.17235 \end{bmatrix} \begin{matrix} 2 \\ 3 \end{matrix}$$

Global force vector

$$F = \begin{bmatrix} 0.24129 \\ 1000.41364 \\ 0.17235 \end{bmatrix} \begin{matrix} 1 \\ 2 \\ 3 \end{matrix}$$

Equilibrium eqn  $[K][q] = [F]$

$$10^5 \begin{bmatrix} 0.875 & -0.875 & 0 \\ -0.875 & 1.5 & -0.625 \\ 0 & -0.625 & 0.625 \end{bmatrix} \begin{bmatrix} q_1 \\ q_2 \\ q_3 \end{bmatrix} = \begin{bmatrix} 0.24129 \\ 1000.41364 \\ 0.17235 \end{bmatrix}$$

Applying B.C :- Using Elimination method

$$10^5 \begin{bmatrix} 0.875 & -0.875 & 0 \\ -0.875 & 1.5 & -0.625 \\ 0 & -0.625 & 0.625 \end{bmatrix} \begin{bmatrix} q_1 \\ q_2 \\ q_3 \end{bmatrix} = \begin{bmatrix} 0.24129 \\ 1000.41364 \\ 0.17235 \end{bmatrix}$$

$$q_1 = 0 ; \quad q_2 = 0.01144 \text{ mm} ; \quad q_3 = 0.01144 \text{ mm}$$

Displacement at Node 1 ;  $q_1 = 0$

Node 2 ;  $q_2 = 0.01144 \text{ mm}$

Node 3 ;  $q_3 = 0.01144 \text{ mm}$

2

## QUADRATIC ELEMENT OR 3 NODED BAR ELEMENT

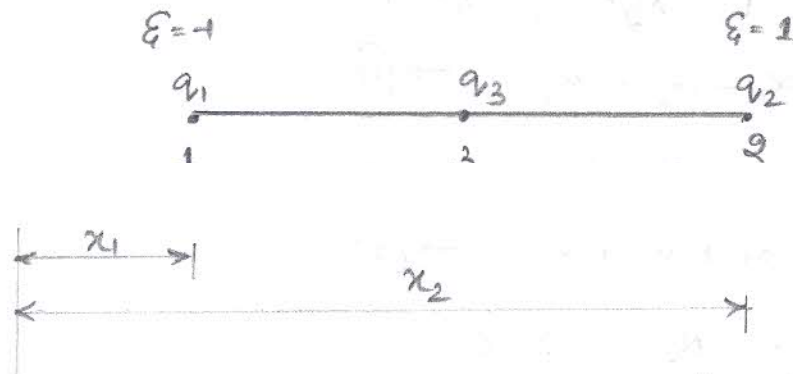


Fig. Shows a 3 noded bar element having end nodes 1, 2 & midside node 3. At each node 1 DoF.

### DERIVATION OF SHAPE FUNCTION

$$\text{Let } N_1 = \alpha_1 + \alpha_2 \xi + \alpha_3 \xi^2$$

$$\text{At node 1, } N_1 = 1, \xi = -1.$$

$$1 = \alpha_1 - \alpha_2 + \alpha_3 \rightarrow (1)$$

$$\text{At node 2, } N_1 = 0, \xi = 1.$$

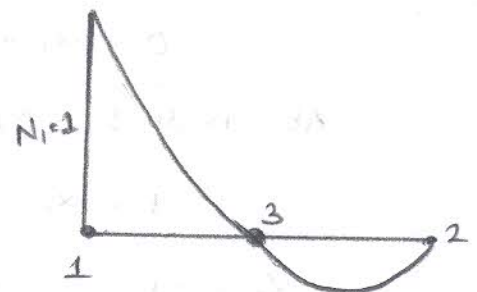
$$0 = \alpha_1 + \alpha_2 + \alpha_3 \rightarrow (2)$$

$$\text{At node 3, } N_1 = 0, \xi = 0.$$

$$\alpha_1 = 0 \rightarrow (3)$$

$$\alpha_2 = -1/2 \quad ; \quad \alpha_3 = 1/2$$

$$N_1 = \frac{\xi}{2} (\xi - 1)$$



$$\text{Let } N_2 = \alpha_1 + \alpha_2 \xi + \alpha_3 \xi^2$$

$$\text{At node 1, } N_2 = 0, \xi = -1.$$

$$0 = \alpha_1 - \alpha_2 + \alpha_3 \rightarrow (4)$$

$$\text{At node 2, } N_2 = 1, \xi = 1$$

$$1 = \alpha_1 + \alpha_2 + \alpha_3 \rightarrow (5)$$

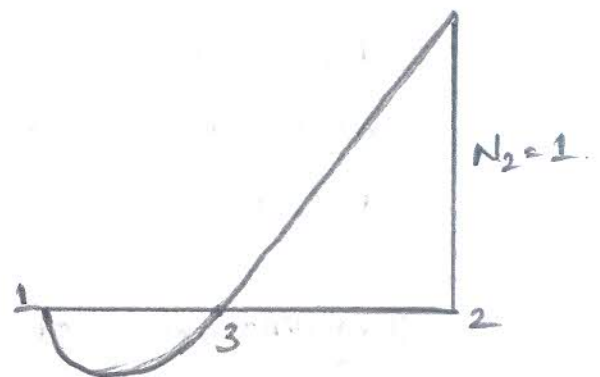
$$\text{At node 3, } N_2 = 0, \xi = 0$$

$$0 = \alpha_1 \rightarrow (6)$$

$$\alpha_2 = 1/2 \quad ; \quad \alpha_3 = 1/2$$

$$N_2 = \frac{1}{2} \xi + \frac{1}{2} \xi^2$$

$$N_2 = \frac{\xi}{2} (\xi + 1)$$



$$\text{Let } N_3 = \alpha_1 + \alpha_2 \xi + \alpha_3 \xi^2$$

$$\text{At node 1, } N_3 = 0, \xi = -1.$$

$$0 = \alpha_1 - \alpha_2 + \alpha_3 \rightarrow (7)$$

$$\text{At node 2, } N_3 = 0, \xi = 1.$$

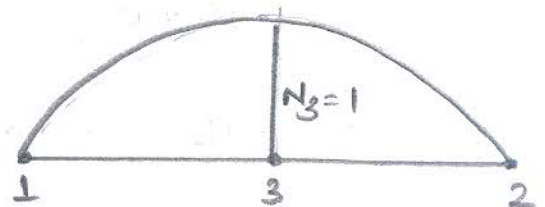
$$0 = \alpha_1 + \alpha_2 + \alpha_3 \rightarrow (8)$$

$$\text{At node 3, } N_3 = 1, \xi = 0$$

$$1 = \alpha_1 \rightarrow (9)$$

$$\alpha_3 = -1, \quad \alpha_2 = 0$$

$$N_3 = 1 - \xi^2$$



3

### DERIVATION OF ELEMENTAL STIFFNESS MATRIX FOR A TRUSS ELEMENT

Relation b/w nodal displacement of a truss element in local coordinates & global coordinate is expressed as

$$q' = L q$$

$$\text{where } q' = \begin{bmatrix} q'_1 \\ q'_2 \end{bmatrix}; \quad L = \begin{bmatrix} l & m & 0 & 0 \\ 0 & 0 & l & m \end{bmatrix}; \quad q = \begin{bmatrix} q_1 \\ q_2 \\ q_3 \\ q_4 \end{bmatrix}$$

A truss element in local coordinate is equivalent to 1D bar element having the stiffness matrix

$$K' = \frac{EA}{l_e} \begin{bmatrix} 1 & -1 \\ -1 & 1 \end{bmatrix}$$

Strain energy for a truss element in local coordinate is given by

$$U_e = \frac{1}{2} q'^T K' q' \rightarrow \textcircled{1}$$

It is required to determine strain energy of truss element in global coordinates.

$$q' = L q$$

Sub. this in eqn  $\textcircled{1}$  we get

$$U_e = \frac{1}{2} q^T L^T K' L q$$



$$U_e = \frac{1}{2} q^T (L^T K' L) q$$

$$U_e = \frac{1}{2} q^T K_e q$$

Where  $K_e$  - elemental stiffness matrix in global coordinates

$$K_e = L^T K' L$$

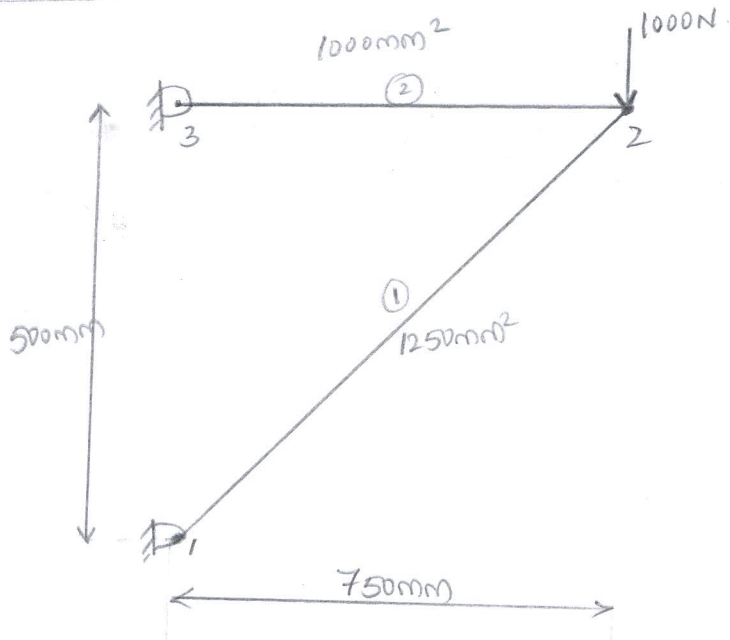
$$L = \begin{bmatrix} l & m & 0 & 0 \\ 0 & 0 & l & m \end{bmatrix}; \quad L^T = \begin{bmatrix} l & 0 \\ m & 0 \\ 0 & l \\ 0 & m \end{bmatrix}; \quad K' = \frac{EA}{l_e} \begin{bmatrix} 1 & -1 \\ -1 & 1 \end{bmatrix}$$

$$K_e = \begin{bmatrix} l & 0 \\ m & 0 \\ 0 & l \\ 0 & m \end{bmatrix} \frac{EA}{l_e} \begin{bmatrix} 1 & -1 \\ -1 & 1 \end{bmatrix} \begin{bmatrix} l & m & 0 & 0 \\ 0 & 0 & l & m \end{bmatrix}$$

$$K_e = \frac{EA}{l_e} \begin{bmatrix} l^2 & lm & -l^2 & -lm \\ lm & m^2 & -lm & -m^2 \\ -l^2 & -lm & l^2 & lm \\ -lm & -m^2 & lm & m^2 \end{bmatrix}$$

Where  $l$  &  $m$  are direction cosines.

4.



$$E_1 = E_2 = 200 \text{ GPa}$$

SolNodal Data

Node No	x (mm)	y (mm)
1	0	0
2	750	500
3	0	500

Element Connectivity table

Element No	Initial Node (IN)	Final Node (FN)	length of element $l_e$	$l$	$m$
1	1	2	901.387	0.832	0.554
2	2	3	750	-1	0

$$l_e = \sqrt{(x_{FN} - x_{IN})^2 + (y_{FN} - y_{IN})^2}$$

$$l = \frac{x_{FN} - x_{IN}}{l_e} ; \quad m = \frac{y_{FN} - y_{IN}}{l_e}$$

$$K = \frac{EA}{le} \begin{bmatrix} l^2 & lm & -l^2 & -lm \\ lm & m^2 & -lm & -m^2 \\ -l^2 & -lm & l^2 & lm \\ -lm & -m^2 & lm & m^2 \end{bmatrix}$$

For element 1

$$K = \frac{2 \times 10^5 \times 1250}{901.387} \begin{bmatrix} 0.69 & 0.46 & -0.69 & -0.46 \\ 0.46 & 0.31 & -0.46 & -0.31 \\ -0.69 & -0.46 & 0.69 & 0.46 \\ -0.46 & -0.31 & 0.46 & 0.31 \end{bmatrix}$$

$$K_1 = 10^5 \begin{bmatrix} 1.9 & 1.27 & -1.9 & -1.27 \\ 1.27 & 0.86 & -1.27 & -0.86 \\ -1.9 & -1.27 & 1.9 & 1.27 \\ -1.27 & -0.86 & 1.27 & 0.86 \end{bmatrix} \begin{matrix} 1 \\ 2 \\ 3 \\ 4 \end{matrix}$$

$$K_2 = 10^5 \begin{bmatrix} 2.66 & 0 & -2.66 & 0 \\ 0 & 0 & 0 & 0 \\ -2.66 & 0 & 2.66 & 0 \\ 0 & 0 & 0 & 0 \end{bmatrix} \begin{matrix} 3 \\ 4 \\ 5 \\ 6 \end{matrix}$$

Global Stiffness

$$K = 10^5 \begin{bmatrix} 1.9 & 1.27 & -1.9 & -1.27 & 0 & 0 \\ 1.27 & 0.86 & -1.27 & 0.86 & 0 & 0 \\ -1.9 & -1.27 & 4.56 & 1.27 & -2.66 & 0 \\ -1.27 & 0.86 & 1.27 & 0.86 & 0 & 0 \\ 0 & 0 & -2.66 & 0 & 2.66 & 0 \\ 0 & 0 & 0 & 0 & 0 & 0 \end{bmatrix} \begin{matrix} 1 \\ 2 \\ 3 \\ 4 \\ 5 \\ 6 \end{matrix}$$

Equilibrium eqn

$$[K] [q] = [F]$$

$$10^5 \begin{bmatrix} 1.9 & 1.27 & -1.9 & -1.27 & 0 & 0 \\ 1.27 & 0.86 & -1.27 & 0.86 & 0 & 0 \\ -1.9 & -1.27 & 4.56 & 1.27 & -2.66 & 0 \\ -1.27 & 0.86 & 1.27 & 0.86 & 0 & 0 \\ 0 & 0 & -2.66 & 0 & 2.66 & 0 \\ 0 & 0 & 0 & 0 & 0 & 0 \end{bmatrix} \begin{bmatrix} q_1 \\ q_2 \\ q_3 \\ q_4 \\ q_5 \\ q_6 \end{bmatrix} = \begin{bmatrix} 0 \\ 0 \\ 0 \\ -1000 \\ 0 \\ 0 \end{bmatrix}$$

Applying B.C. Using elimination method.

$$10^5 \begin{bmatrix} 4.56 & 1.27 \\ 1.27 & 0.86 \end{bmatrix} \begin{bmatrix} q_3 \\ q_4 \end{bmatrix} = \begin{bmatrix} 0 \\ -1000 \end{bmatrix}$$

$$q_3 = -5.5 \times 10^{-3} \text{ mm}$$

$$q_4 = 0.01975 \text{ mm}$$

Nodal displacements.

$$\text{At nodes 1, 3} = q_1 = q_2 = q_5 = q_6 = 0.$$

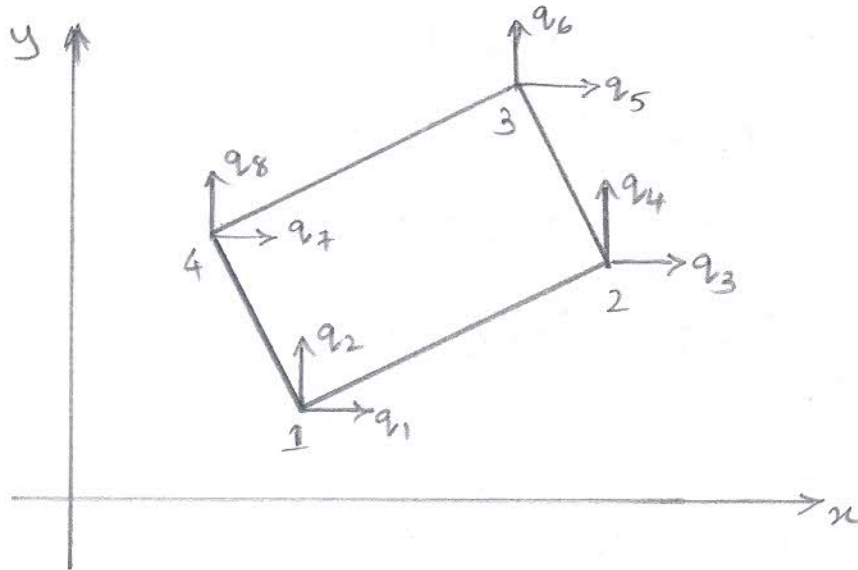
$$\text{At node 2 ; } q_3 = -5.5 \times 10^{-3} \text{ mm} //$$

$$q_4 = 0.01975 \text{ mm} //$$

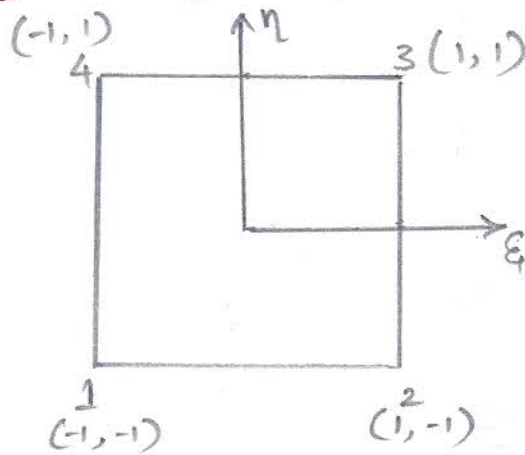
5

## 4 NODED QUADRILATERAL ELEMENT [QUAD 4 ELEMENT]

Fig. shows a quadrilateral element having 4 Nodes at each node 2 DOF.



## DERIVATION OF SHAPE FUNCTION



$$N_i = \frac{1}{4} (1 + \xi \xi_i) (1 + \eta \eta_i)$$

$$i = 1, 2, 3, 4$$

$$N_1 = \frac{1}{4} (1 - \xi) (1 - \eta)$$

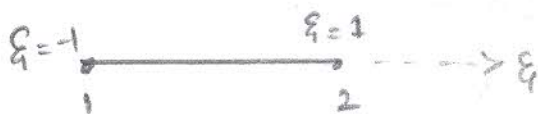
$$N_2 = \frac{1}{4} (1 + \xi) (1 - \eta)$$

$$N_3 = \frac{1}{4} (1 + \xi) (1 + \eta)$$

$$N_4 = \frac{1}{4} (1 - \xi) (1 + \eta)$$

### LAGRANGEAN METHOD

$$N_i(\xi, \eta) = L_i(\xi) L_i(\eta)$$



$$L_1(\xi) = \frac{\xi - \xi_2}{\xi_1 - \xi_2} = \frac{\xi - 1}{-1 - 1}$$

$$L_1(\xi) = \frac{1 - \xi}{2}$$

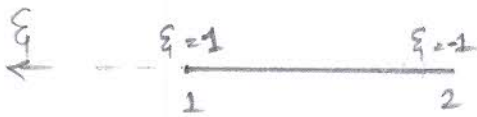


$$L_1(\eta) = \frac{\eta - \eta_4}{\eta_1 - \eta_4} = \frac{\eta - 1}{-1 - 1} = \frac{1 - \eta}{2}$$

$$\begin{aligned} N_1 &= L_1(\xi) L_1(\eta) \\ &= \frac{1 - \xi}{2} \cdot \frac{1 - \eta}{2} \end{aligned}$$

$$N_1 = \frac{1}{4} (1 - \xi)(1 - \eta)$$

$$N_2(\xi, \eta) = L_2(\xi) L_2(\eta)$$



$$L_2(\xi) = \frac{\xi - \xi_1}{\xi_2 - \xi_1} = \frac{\xi - 1}{-1 - 1} = \frac{1 + \xi}{2}$$



$$L_2(\eta) = \frac{\eta - \eta_3}{\eta_2 - \eta_3} = \frac{\eta + 1}{-1 - 1} = \frac{1 - \eta}{2}$$

$$N_2 = \frac{1 + \xi}{2} \cdot \frac{1 - \eta}{2}$$

$$N_2 = \frac{1}{4} (1 + \xi)(1 - \eta)$$

$$N_3(\xi, \eta) = L_3(\xi) L_3(\eta)$$



$$L_3(\xi) = \frac{\xi - \xi_4}{\xi_3 - \xi_4} = \frac{\xi + 1}{+1 + 1} = \frac{1 + \xi}{2}$$



$$L_3(\eta) = \frac{\eta - \eta_2}{\eta_3 - \eta_2} = \frac{\eta + 1}{1 + 1} = \frac{1 + \eta}{2}$$

$$N_3 = \frac{1}{4} (1+\xi) (1+\eta)$$

$$N_4(\xi, \eta) = L_4(\xi) L_4(\eta)$$



$$L_4(\xi) = \frac{\xi - \xi_3}{\xi_4 - \xi_3} = \frac{\xi - 1}{-1 - 1} = \frac{1 - \xi}{2}$$

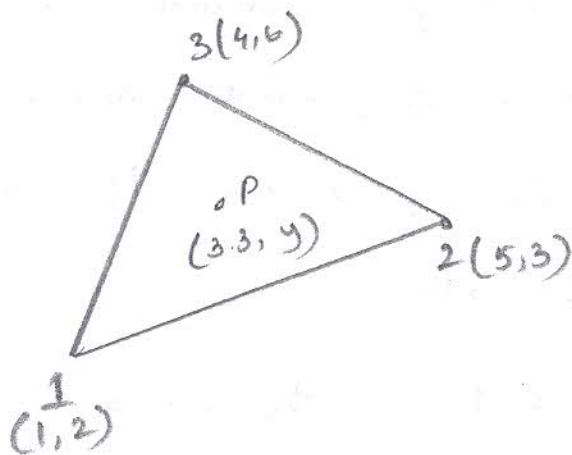


$$L_4(\eta) = \frac{\eta - \eta_1}{\eta_4 - \eta_1} = \frac{\eta + 1}{1 + 1} = \frac{1 + \eta}{2}$$

$$N_4 = \frac{1}{4} (1 - \xi) (1 + \eta)$$



6 a



$$x = N_1 x_1 + N_2 x_2 + N_3 x_3$$

$$3.3 = 0.3(1) + \eta(5) + (1 - \xi - \eta)4$$

$$3.3 = 0.3 + 5\eta + 4 - 4\xi - 4\eta$$

$$-4\xi + \eta = -1$$

$$-4(0.3) + \eta = -1$$

$$\eta = 0.2$$

$$\therefore N_2 = \eta = 0.2 //$$

$$N_3 = 1 - \xi - \eta = 1 - 0.3 - 0.2 = 0.5$$

$$N_3 = 0.5 //$$

y coordinate

$$y = N_1 y_1 + N_2 y_2 + N_3 y_3$$

$$= 0.3(2) + 0.2(3) + 0.5(6)$$

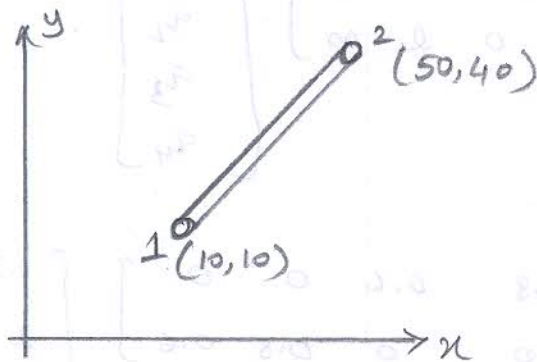
$$y = 4.2 //$$

6.b

For the truss element shown  $(x, y)$  coordinates are indicated near nodes 1, 2. The <sup>element</sup> displacement dof vector is given by  $\{q\} = [1.5, 1.0, 2.1, 4.3]^T \times 10^{-2} \text{ mm}$ .

Take  $E = 300 \times 10^3 \text{ N/mm}^2$ .  $A = 100 \text{ mm}^2$ . Determine

- Element displacement dof in local coordinate  $(q'_1, q'_2)$
- Stress in element
- Stiffness matrix of element



Node Data

Node No	$x$ (mm)	$y$ (mm)
1	10	10
2	50	40

Element Connectivity Table

Element No	Initial Node	Final Node	Length of element	$l$	$m$
1	1	2	50	0.8	0.6

$$l_e = \sqrt{(x_2 - x_1)^2 + (y_2 - y_1)^2} = \sqrt{(50 - 10)^2 + (40 - 10)^2}$$

$$l_e = 50 \text{ mm}$$

Sol

$$l = \frac{x_{FN} - x_{IN}}{l_e} = \frac{40}{50} = 0.8$$

$$m = \frac{y_{FN} - y_{IN}}{l_e} = \frac{30}{50} = 0.6$$

i) Element displacement dof in local coordinate

$$a' = L q$$

$$\begin{bmatrix} a'_1 \\ a'_2 \end{bmatrix} = \begin{bmatrix} l & m & 0 & 0 \\ 0 & 0 & l & m \end{bmatrix} \begin{bmatrix} q_1 \\ q_2 \\ q_3 \\ q_4 \end{bmatrix}$$

$$= \begin{bmatrix} 0.8 & 0.6 & 0 & 0 \\ 0 & 0 & 0.8 & 0.6 \end{bmatrix} \begin{bmatrix} 1.5 \\ 1 \\ 2.1 \\ 4.3 \end{bmatrix} \times 10^{-2}$$

$$\begin{bmatrix} a'_1 \\ a'_2 \end{bmatrix} = \begin{bmatrix} 0.018 \\ 0.0426 \end{bmatrix} \text{ mm} //$$

ii) Stress  $\sigma = E \cdot \frac{1}{l_e} [-l \quad -m \quad l \quad m] \begin{bmatrix} q_1 \\ q_2 \\ q_3 \\ q_4 \end{bmatrix}$

$$= 300 \times 10^3 \times \frac{1}{50} [-0.8 \quad -0.6 \quad 0.8 \quad 0.6] \begin{bmatrix} 1.5 \\ 1 \\ 2.1 \\ 4.3 \end{bmatrix} \times 10^{-2}$$

$$\sigma = 147.6 \text{ N/mm}^2 //$$

## Stiffness matrix

$$K = \frac{EA}{le} \begin{bmatrix} l^2 & lm & -l^2 & -lm \\ lm & m^2 & -lm & -m^2 \\ -l^2 & -lm & l^2 & lm \\ -lm & -m^2 & lm & m^2 \end{bmatrix}$$

$$= \frac{300 \times 10^3 \times 100}{50} \begin{bmatrix} 0.64 & 0.48 & -0.64 & -0.48 \\ 0.48 & 0.36 & -0.48 & -0.36 \\ -0.64 & -0.48 & 0.64 & 0.48 \\ -0.48 & -0.36 & 0.48 & 0.36 \end{bmatrix}$$

$$K = 10^5 \begin{bmatrix} 3.84 & 2.88 & -3.84 & -2.88 \\ 2.88 & 2.16 & -2.88 & -2.16 \\ -3.84 & -2.88 & 3.84 & 2.88 \\ -2.88 & -2.16 & 2.88 & 2.16 \end{bmatrix} //$$