



## Internal Assessment Test 3 – Dec. 2022

Sub:	Design of RC and Steel Structure					Sub Code:	18CV72	Branch		Civi	Civil	
Date:	27/12/2022 Duration: 90 min's Max Marks: 50 Sem /					Sem / Sec:	7A			О	OBE	
Answer ONE FULL Questions Note: Use of IS 456:2000 is permitted. Assume missing data suitably.									MARI S	CO	RB T	
1	Design a welded plate girder for an effective span of 14 m. Imposed load on the girder consist of UDL 45 kN/m in addition to 2-point loads each of magnitude 400kN, placed at 3m on either side of the mid span point of the girder. Design i. Mid span cross section, ii. Curtailment of flange plates, iii. Intermediate stiffeners, iv. End bearing and bearing stiffeners								[50]	CO3	L4	
Or												
2	Design a gantry girder to be used in an industrial building carrying an electrically operated overhead travelling crane, for the following data • Centre to Centre between distance between gantry rails or span of crane girders 25 m, Centre to Centre distance between columns (span of gantry girder) 8 m, Crane capacity 200 kN • Self-weight of the crane girder excluding trolley 150 kN at centre or (150/25 = 6kN/m), • Self-weight of the crab or trolley, electric motor, hook, etc. 75 kN, Approximate minimum approach of the crane hook to the gantry girder 1.0 m, Wheel base 3.5 m, Self-weight of rail section 300 N/m =0.3kN/m, Diameter of crane wheels 150 mm, Height of rail 105 mm, Steel is of grade Fe 410. Design also the field welded connection if required. The support bracket connection need not be designed.							[50]	CO2	L2		

Paper 3
CI CCI HOD

For Plastic & Compact Condition

(b) < 3.4 \( \text{ } \) < 9.4 \( \text{ } \) 

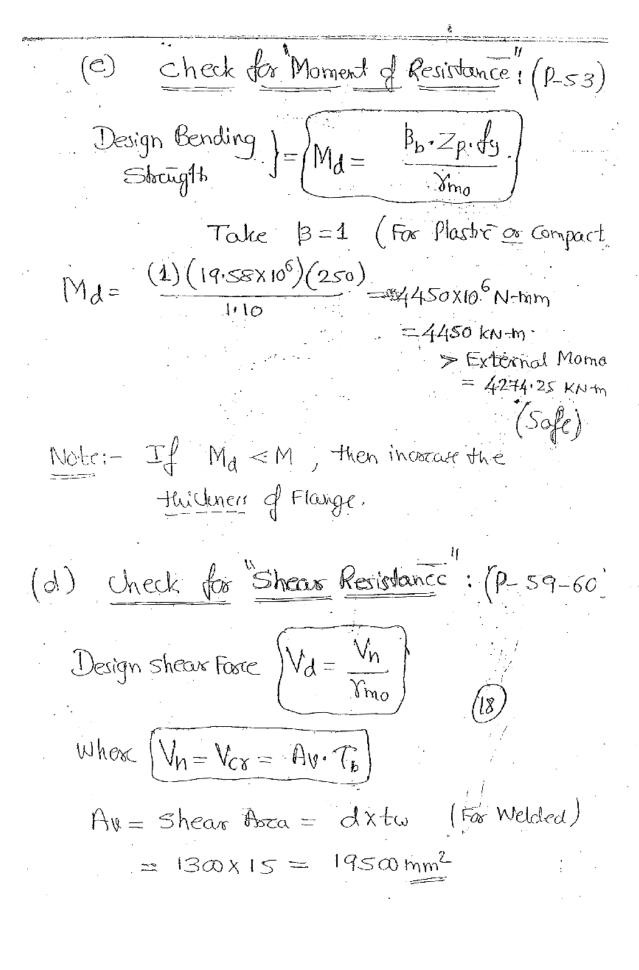
Vsive 
$$\left( \frac{b}{tf} \right) = 8.4 \( \frac{192.5}{8.4 \times 1} \right) = tf$$

if = 22.92 mm

Provide Flange — bf x tf = 400 mm x 25 mm

 $\left( \frac{400 \text{ mm}}{192.5 \text{ mm}} \right) = \frac{400 \text{ mm}}{120.5 \text{ mm}} = \frac{400 \text{ m$ 

Zp=19.58X10 mm3





$$V_{C8} = A_{8} \times T_{b} = 19500 \times 130.48$$
  
= 2544.3 × 10<sup>3</sup> M

:. Design Shear 
$$J = V_d = \frac{V_{cs}}{V_{mo}} = \frac{2544.3}{1110}$$

$$V_d = 2313.0 \, \text{kN} \Rightarrow \text{External S.F}$$

$$= 1135.5 \, \text{kN}$$

$$\begin{array}{c|c}
\hline
400mm \\
\hline
9 & 650 \\
\hline
25mm \\
\hline
2 & 7
\end{array}$$

$$F = \frac{(1135.5 \times 10^3)(400 \times 25)(650 + \frac{25}{2})}{1.152 \times 10^{10}} = \frac{(653.0 \text{ N/mm})}{(653.0 \text{ N/mm})}$$

Face per mm length.

Equasting above force with strongth of weld per mm length. 653 N/mm = (0.75) (1) ( fa / V3 8mw) 2.  $= \left[ \left( 0.7 \times S \right) \left( 1 \right) \frac{410}{\sqrt{3} \times 1.25} \right]$ : Sixed Weld S = 2:46 , welding. Provide Min. Size S = 5mm (2) Design of Intermediate tw=15mm specing 'c'

(i) Radio ->  $\left(\frac{d}{t\omega}\right) = \left(\frac{1300}{15}\right) = 86.67 > 67$ Hence provide I.S. { The radio should not } exceed 67



Provide I.S. of Size 80mm x 8mm on either side of Web. (IV) Connection of I.S. to the Web: Force required for  $J = \left(\frac{+\omega^2}{5bs^2}\right)$  lage 67 the Connection  $\frac{1}{5bs^2}$  kn/m tw = 15mm, bs = Stiffener out standing  $\frac{(15)^2}{5 \times 80} = 0.563 \, \text{kn/mm}$ = (563 N/mm) -> Equating to the strongth of the Weld permin 563 N/mm = TO.7XSXXX Fu 4 Four side  $= \int_{0.7}^{0.7} x \, dx \, 4mm \, x \, \frac{410}{\sqrt{3} \times 1.25} \, x \, 4$ Provide Min. size of Weld = 3mm, - Continuous Weld.

## (3) Curtailment of Flange Plate:

and it decreases towards the support. Hence, from the economical point of view decrease the thickness of large flate as moment dureases called curtailine. of flarge plate. Standard Thibaness of Plate 6, 20, 22, 25. 28, 30, 32, 40

Reduce the thickness of Flange plate from 25mm to 16mm

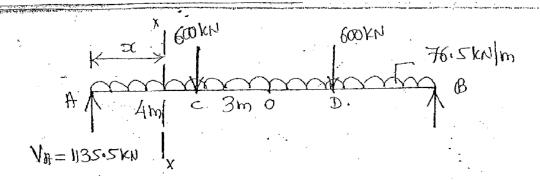
Plastic Modulus 
$$Zp = \Sigma ay$$

$$Zp = 2 \left[ (400 \times 16) (650 + \frac{16}{2}) + (15 \times 650) \frac{650}{2} \right]$$

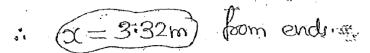
Design Bending 
$$= M_d = \frac{B_b \cdot Z_p \cdot f_y}{8mo}$$
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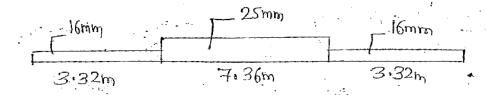
$$= \frac{(1)(14.76\times10^{6})250}{1.10} = 3355\times10^{6} \text{ N-mm}$$

$$= (3355 \text{ kN-m})$$



 $M_{XX} = 1135.50c - 76.5 \times 0.000 = 33.55 \text{ km-m}$ 



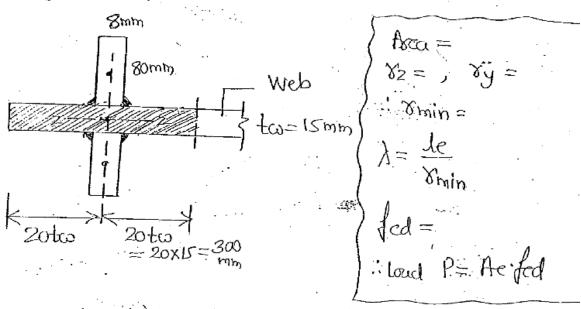


HW Try One more Curtailment

Reduce from 25mm to 20mm

Calculate Zp, Md & X

Let us provide E.B.S. plates of size 80mm x 8mm
(Assumed)



$$I_{ZZ} = 2\left[\frac{8\times80}{12} + 60\times15 = 10280 \text{ mm}^2\right] + \left[\frac{8\times80}{12} + (8\times80)\left(\frac{80}{2} + \frac{15}{2}\right)^2\right] + \left[\frac{60\times15}{12} + 0\right] = 3.74\times16^6 \text{ mm}^4$$

$$\therefore \text{ 8min} = \sqrt{\frac{I_{min}}{A}} = \sqrt{\frac{3.74\times106}{10280}} = 19.07 \text{ mm}$$

$$\lambda = \frac{Je}{S_{min}} = \frac{0.7d}{S_{min}} = \frac{0.7 \times 1300}{19.07} = [47.72]$$

From Table 9(c) -> fcd = 186.42 N/mm2

(Sak)

(III) Connection:
Force per mm height ) = Reaction = 1135.5×103N  g with 1 = Reaction = 1135.5×103N
= [873.46 N/mm] ->
Equating the above face with straight fixed
$873.46  \text{N/mm} = 4 \left(0.7 \times \text{S}\right) \left(1 \text{mm}\right) \frac{410}{\sqrt{3} \times 1.25}$
: 5 = 1.64mm.
Provide Min. Sije of Weld = 3mm
——————————————————————————————————————

< Reachion = 1135.5 kg

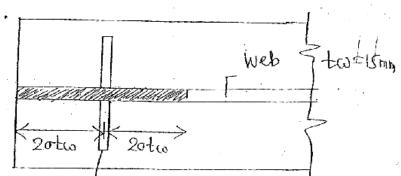
Hence provide E.B.S.

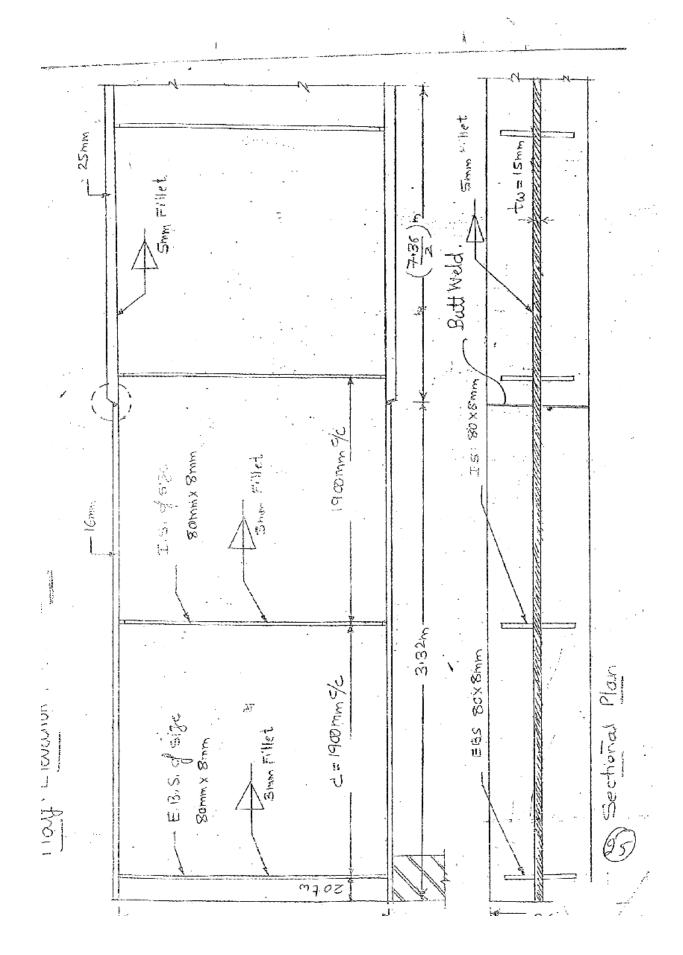
$$\frac{Aq \times 250}{0.8 \times 1.10} = 1135.5 \times 10^{3}$$

$$Aq = 3997 \text{ mm}^{2} \approx 4000 \text{ mm}^{2}$$

The E.B.S. and designed like a compression member subjected to reaction. and effective height is "0.7d."

Along with E.B.S. plates, the part of the web (20tw on either side) also carry reactions.





50/2 (a) Load calculation

Max. ocachion in coane boidge occurs when the toolley along with book if it is towards left. or night with a minimum hook distance Im.

"Lader take moment under the wheel load which is very close to mid span. ≥ MD=0, RCX8-212x 1,375-212(1,375+1,75+1,75)= Rc = 165.63 KN ... : Mmax) x = Rcx 3.125 = 165.63 x 3.125 = 517.6k : Factored Moment M = 776.4 KN-m Max. Shear Force: The arrangement of wheel land for Max. SF is, Two wheel loads is placed either complete left or sight dide of the span."

≥ MD =0 RCX8-212x45-212x8=0 : Vmax = Rc = 331.25 kN : Factored SF V = 496.88 KN (c) Honjowtal load of Its Moment , A lateral load le developed due to the application of brakes or acceleration of the trolley. It is taken 10%. of lifted weight and thalley neight. Horizantal Force = 10 (200+75) . H wheels = 16.875KN = 7KN factored force = 10.5 kn -\*Moment Die to ] = 7.76.4 ×10.5 = 25.64 kp-m] 1 lad factor.

6



I\_ = 4765×109mm4 Increase the above value by 30% cyproximately  $= 1.30 \times 1.765 \times 10^9 = 2.295 \times 10^9 \, \text{mm}^4$ = 229 500 cm4 From steel Table Try suitable section. ISWB-500@95.2kg/m ISMC-400 @ 49.4 kg/m Top cover plade -> 320x 20mm Bothom - " - 320x40mm Total Iz=Ix= 230 1945 cm4 \_tw=8.6mm 234.99 m EAA 568.6mm 235.61 = h

## Overall Properties Ixx = Izz = 230194.5x104mm4 Arca = 376.15x100 mm²

$$Cxx = 283.7 \text{mm} (750 \text{m Top}) \text{ } Cxx = 284.9 \text{mm} (800 \text{m})$$
  
 $8yy = 9.57 \text{cm} = 95.7 \text{mm}$ 

Arta of Shaded Portion = 
$$\frac{1}{2}$$
 (Total Arta)

$$(9.9 \times h) + (250 \times 14.7) + (320 \times 40) = \frac{1}{2} [376.15 \times 100]$$

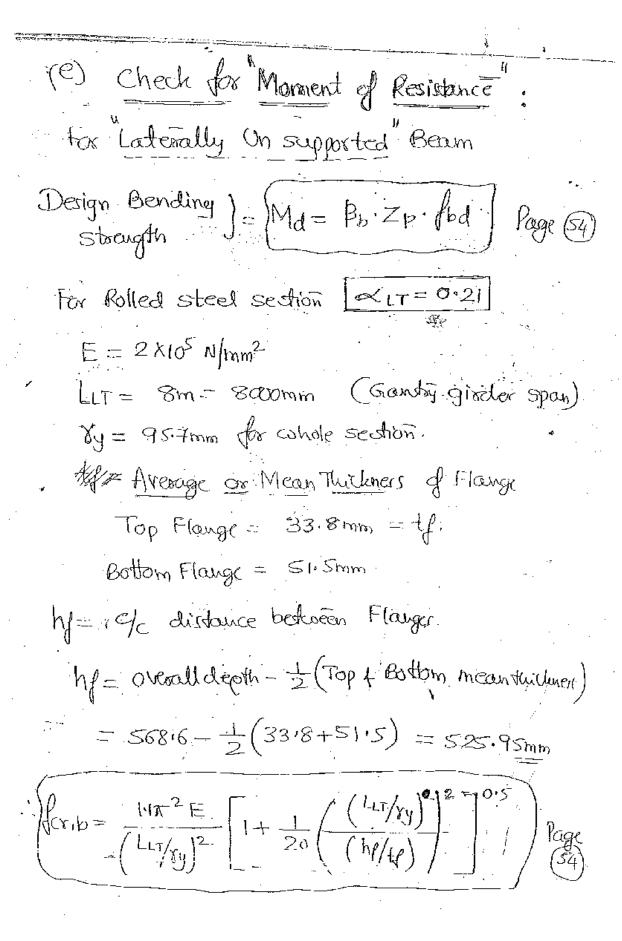
$$n = 235.61 \text{ mm}$$

$$Zp = (320 \times 40)(235.61 + 14.7 + \frac{40}{2}) + (250 \times 14.7)(235.61 + \frac{14.7}{2})$$

$$+(9.9 \times 235.61) \frac{235.61}{2} + (234.99 \times 9.9) \frac{234.99}{2}$$

+ 
$$(250 \times 14.7)(234.99 + \frac{14.7}{2}) + (320 \times 20)(234.99 + 14.7 + \frac{20}{2})$$

$$f = (6293)(278.29 - 24.2) = 9.05 \times 10^6 \text{ mm}^3$$



$$\frac{\left(\cos b = \frac{1.1 \times 22\times10^{5}}{(\frac{8000}{95.7})^{2}} \left[1 + \frac{1}{20} \left(\frac{8000/95.7}{525.95/33.8}\right)^{2}\right] 0.5}{\left(\frac{8000}{95.7}\right)^{2}} = \frac{1.1 \times 22\times10^{5}}{(\frac{8000}{95.7})^{2}} \left[1 + \frac{1}{20} \left(\frac{8000/95.7}{525.95/33.8}\right)^{2}\right] 0.5}$$

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 $M_d = 1701.3 \times 10^6 N - mm > 776.4 km - m$ 

(Safe).

Design Shear Strongth Vd = 
$$\frac{V_n}{8mo} = \frac{A_W \cdot f_{9W}}{\sqrt{3} \times 8mo}$$
P-59

An = Shear area = hxto For tot Rolled.

h = overall depth of the section (31)

 $A_0 = 568.6 \times 9.9 = 5629.14$ 

$$V_{d} = \frac{(5629.14) \times 250}{\sqrt{3} \times 1.10} = 738.63 \times 10^{3} \text{N}$$

$$> 496.88 \text{ km} \text{ Soft}$$

$$(SF) \cdot \text{Soft}$$

N2= 136.75mm

(Es

$$F_{\omega} = (100 + 136.75) \times 9.9 \times 250$$

$$F_{\omega} = 532.68 \text{ km} > 496.88 \text{ km}$$

$$(S.F)$$

