

Internal Assessment Test III – July 2023

Sub: Finite Element Methods
Date: 04/07/2023 **Duration:** 90 mins **Max Marks:** 50 **Sem:** VI
Note: Answer all questions.

Code: 18ME61
Branch: MECH

Marks OBE
CO RBT
06 CO4 L2

- Briefly explain the different types of boundary conditions in heat transfer analysis
- A composite wall is as shown in figure 1 consists of three materials. The outer temperature T_o is 20°C . Convective heat transfer takes place on the inner surface of the wall with $T_\infty = 800^\circ\text{C}$. The convective heat transfer coefficient is $25 \text{ W/m}^2 \text{ }^\circ\text{C}$. Determine the temperature distribution in the wall.

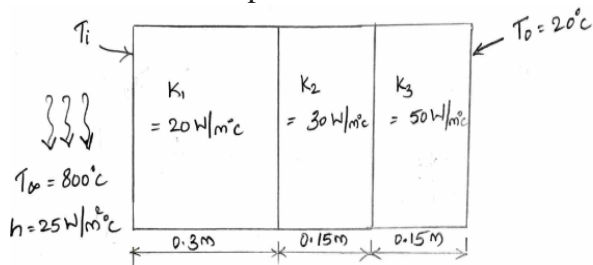


Figure 1

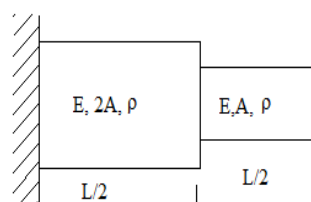


Figure 2

14 CO4 L2

- Derive consistent mass matrix for truss element.
- Evaluate Eigen values & Eigen vector for stepped bar shown in figure 2 when it is subjected to axial vibration, with fixed free end condition as shown. Draw mode shapes

10 CO5 L3
20 CO5 L3

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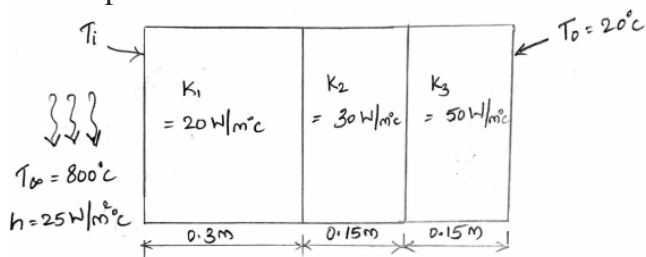


Figure 1

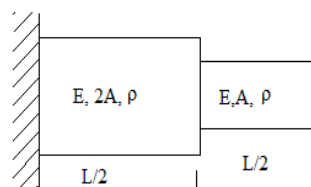


Figure 2

14 CO4 L2

- Derive consistent mass matrix for truss element.
- Evaluate Eigen values & Eigen vector for stepped bar shown in figure 2 when it is subjected to axial vibration, with fixed free end condition as shown. Draw mode shapes

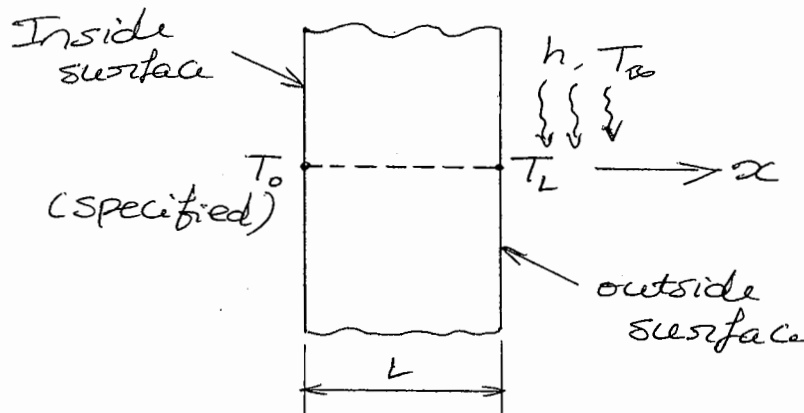
10 CO5 L3
20 CO5 L3

Types of boundary conditions:-

The boundary conditions are mainly of three kinds.

- 1) specified temperature boundary condition
- 2) specified heat flux (Insulated) boundary cond?
- 3) Convection boundary condition.

For example consider the wall of a tank as shown in the figure below. The tank contains a hot liquid at a temperature ' T_0 ' with an airstream of temperature T_∞ passed on the outside maintaining a wall temperature of ' T_L ' at the boundary.

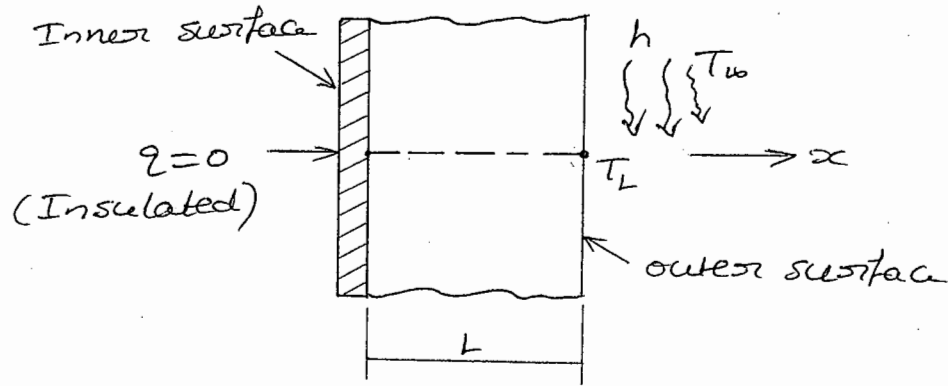


The boundary condition for this problem is

$$\text{At } x=0, \quad T=T_0$$

$$\text{At } x=L, \quad q = h(T_L - T_\infty)$$

As another example, consider a wall as shown in the figure below, where the inside surface is insulated & the outside is a convective surface.



The boundary condition for this problem

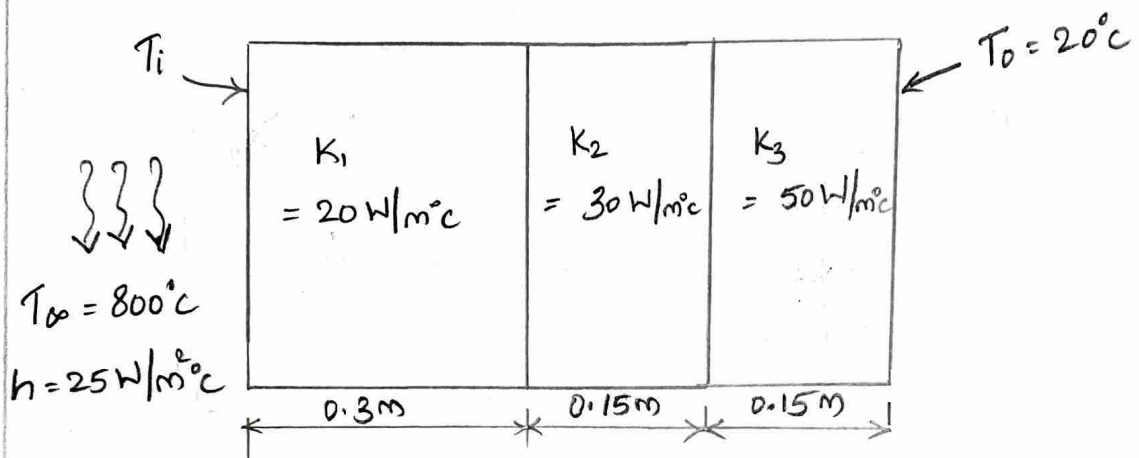
is

At $x=0$, heat flux $q=0$

At $x=L$ heat flux $q=h(T_L - T_\infty)$

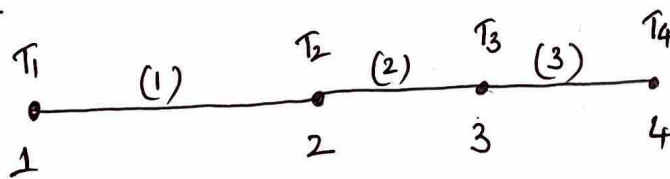
2

A Composite wall is as shown in figure Consists of three materials. The outer temperature T_o is 20°C . Convective heat transfer takes place on inner surface of the wall with $T_\infty = 800^\circ\text{C}$. The Convective heat transfer coefficient is $25\text{ W/m}^2\text{C}$. Determine the temperature distribution in the wall.

Sol

$$k_1 = 20\text{ W/mC} ; k_2 = 30\text{ W/mC} ; k_3 = 50\text{ W/mC}$$

$$h = 25\text{ W/m}^2\text{C} ; T_\infty = 800^\circ\text{C}$$

F.E. ModelElement stiffness matrix

$$[K] = \frac{AK}{L} \begin{bmatrix} 1 & -1 \\ -1 & 1 \end{bmatrix}$$

For element 1

$$[K_1] = \frac{1 \times 20}{0.3} \begin{bmatrix} 1 & -1 \\ -1 & 1 \end{bmatrix} = \begin{bmatrix} 66.67 & -66.67 \\ -66.67 & 66.67 \end{bmatrix}$$

For element 2

$$K_2 = \frac{1(30)}{0.15} \begin{bmatrix} 1 & -1 \\ -1 & 1 \end{bmatrix} = \begin{bmatrix} 200 & -200 \\ -200 & 200 \end{bmatrix} \begin{matrix} 2 & 3 \\ 3 & 2 \end{matrix}$$

For element 3

$$K_3 = \frac{1(50)}{0.15} \begin{bmatrix} 1 & -1 \\ -1 & 1 \end{bmatrix} = \begin{bmatrix} 333.33 & -333.33 \\ -333.33 & 333.33 \end{bmatrix} \begin{matrix} 3 & 4 \\ 4 & 3 \end{matrix}$$

Global stiffness

$$[K] = \begin{bmatrix} 66.67 & -66.67 & 0 & 0 \\ -66.67 & 266.67 & -200 & 0 \\ 0 & -200 & 533.33 & -333.33 \\ 0 & 0 & -333.33 & 333.33 \end{bmatrix} \begin{matrix} 1 & 2 & 3 & 4 \\ 2 & 1 & 4 & 3 \\ 3 & 4 & 1 & 2 \\ 4 & 3 & 2 & 1 \end{matrix}$$

At node 1 ; Convection is taking place

$$[K_h] = hA \begin{bmatrix} 1 & 0 \\ 0 & 0 \end{bmatrix} = 25(1) \begin{bmatrix} 1 & 0 \\ 0 & 0 \end{bmatrix}$$

$$= \begin{bmatrix} 25 & 0 \\ 0 & 0 \end{bmatrix} \begin{matrix} 1 & 2 \\ 2 & 1 \end{matrix}$$

Global stiffness

$$[K] = [K] + [K_h]$$

$$[K] = \begin{bmatrix} 91.67 & -66.67 & 0 & 0 \\ -66.67 & 266.67 & -200 & 0 \\ 0 & -200 & 533.33 & -333.33 \\ 0 & 0 & -333.33 & 333.33 \end{bmatrix}$$

Force Vector

$$[F_h] = A h T_{\infty} \begin{bmatrix} 1 \\ 0 \end{bmatrix}$$
$$= 1 (25) (800) \begin{bmatrix} 1 \\ 0 \end{bmatrix}$$

$$= \begin{bmatrix} 20,000 \\ 0 \end{bmatrix}$$

Equilibrium eqn $[K][T] = [F]$

$$\begin{bmatrix} 91.67 & -66.67 & 0 & 0 \\ -66.67 & 266.67 & -200 & 0 \\ 0 & -200 & 533.33 & -333.33 \\ 0 & 0 & -333.33 & 333.33 \end{bmatrix} \begin{bmatrix} T_1 \\ T_2 \\ T_3 \\ T_4 \end{bmatrix} = \begin{bmatrix} 20000 \\ 0 \\ 0 \\ 0 \end{bmatrix}$$

Since Temp at node 4 ; $T_4 = 20^\circ\text{C}$

$$-200T_2 + 533.33T_3 - 333.33T_4 = 0$$

$$-200(T_2) + 533.33T_3 - 333.33(20) = 0$$

$$-200T_2 + 533.33T_3 = 6666.67$$

$$\therefore \begin{bmatrix} 91.67 & -66.67 & 0 \\ -66.67 & 266.67 & -200 \\ 0 & -200 & 533.33 \end{bmatrix} \begin{bmatrix} T_1 \\ T_2 \\ T_3 \end{bmatrix} = \begin{bmatrix} 20000 \\ 0 \\ 6666.67 \end{bmatrix}$$

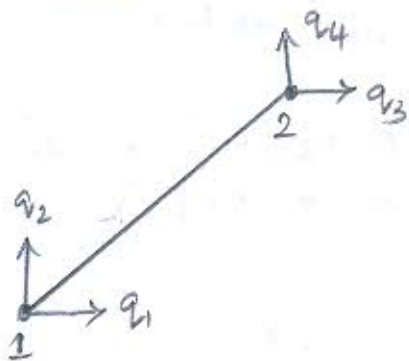
$$T_1 = 304.76^\circ\text{C} ; T_2 = 119.05^\circ\text{C} ; T_3 = 57.14^\circ\text{C}$$

Nodal temp.

$$\begin{bmatrix} T_1 \\ T_2 \\ T_3 \end{bmatrix} = \begin{bmatrix} 304.76 \\ 119.05 \\ 57.14 \end{bmatrix}^\circ\text{C}$$

3

For truss element



Kinetic Energy for a system

$$= \frac{1}{2} \times \text{mass} \times (\text{velocity})^2$$

$$= \frac{1}{2} \times \text{density} \times \text{volume} \times (\text{velocity})^2$$

4

$$\begin{aligned}
&= \frac{1}{2} \int_V \rho \, dv \cdot \dot{u}^T \cdot \dot{u} \\
&= \frac{1}{2} \int_V \rho \, dv \cdot [N \dot{q}]^T N \dot{q} \\
&= \frac{1}{2} \int_V \rho \cdot \dot{q}^T N^T N \dot{q} \, dv \\
&= \frac{1}{2} \int_{le} \rho \dot{q}^T N^T N \dot{q} A \, dx \\
&= \frac{1}{2} \int_{-1}^{+1} \rho \dot{q}^T N^T N \dot{q} A \frac{le}{2} \, d\xi \\
&= \frac{1}{2} \dot{q}^T \left[\frac{\rho A le}{2} \int_{-1}^{+1} N^T N \, d\xi \right] \dot{q} \\
&= \frac{1}{2} \dot{q}^T m^e \dot{q}
\end{aligned}$$

where $m^e \rightarrow$ mass matrix

$$N = \begin{bmatrix} N_1 & 0 & N_2 & 0 \\ 0 & N_1 & 0 & N_2 \end{bmatrix}; \quad N^T = \begin{bmatrix} N_1 & 0 \\ 0 & N_1 \\ N_2 & 0 \\ 0 & N_2 \end{bmatrix}$$

$$m^e = \frac{\rho A le}{2} \int_{-1}^{+1} N^T N \, d\xi$$

$$M^e = \frac{SAle}{2} \int_0^1 \begin{bmatrix} N_1 & 0 \\ 0 & N_1 \\ N_2 & 0 \\ 0 & N_2 \end{bmatrix} \begin{bmatrix} N_1 & 0 & N_2 & 0 \\ 0 & N_1 & 0 & N_2 \end{bmatrix} d\xi$$

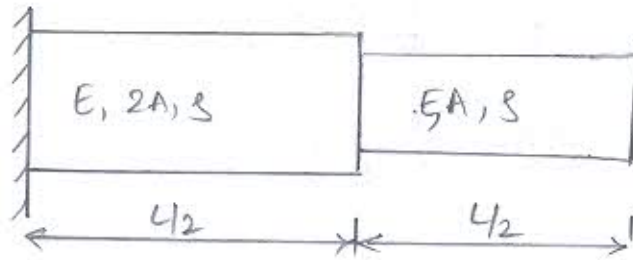
$$= \frac{SAle}{2} \int_0^1 \begin{bmatrix} N_1^2 & 0 & N_1 N_2 & 0 \\ 0 & N_1^2 & 0 & N_1 N_2 \\ N_1 N_2 & 0 & N_2^2 & 0 \\ 0 & N_1 N_2 & 0 & N_2^2 \end{bmatrix} d\xi$$

$$= \frac{SAle}{2} \begin{bmatrix} 2/3 & 0 & 1/3 & 0 \\ 0 & 2/3 & 0 & 1/3 \\ 1/3 & 0 & 2/3 & 0 \\ 0 & 1/3 & 0 & 2/3 \end{bmatrix}$$

$$= \frac{SAle}{6} \begin{bmatrix} 2 & 0 & 1 & 0 \\ 0 & 2 & 0 & 1 \\ 1 & 0 & 2 & 0 \\ 0 & 1 & 0 & 2 \end{bmatrix} //$$

4

Find eigen values & eigen vectors for stepped bar when it is subjected to axial vibration, with fixed free end condition as shown. Draw mode shape.



Sol F.E. Model



Elemental stiffness matrix

$$k = \frac{EA}{L_e} \begin{bmatrix} 1 & -1 \\ -1 & 1 \end{bmatrix}$$

$$k_1 = \frac{4EA}{L} \begin{bmatrix} 1 & -1 \\ -1 & 1 \end{bmatrix} = \frac{EA}{L} \begin{bmatrix} 4 & -4 \\ -4 & 4 \end{bmatrix}_{1,2}^{1,2}$$

$$k_2 = \frac{2EA}{L} \begin{bmatrix} 1 & -1 \\ -1 & 1 \end{bmatrix} = \frac{EA}{L} \begin{bmatrix} 2 & -2 \\ -2 & 2 \end{bmatrix}_{2,3}^{2,3}$$

Global stiffness

$$K = \frac{EA}{L} \begin{bmatrix} 4 & -4 & 0 \\ -4 & 6 & -2 \\ 0 & -2 & 2 \end{bmatrix}_{1,2,3}^{1,2,3}$$

Elemental Mass matrix

$$m^e = \frac{\rho A L e}{6} \begin{bmatrix} 2 & 1 \\ 1 & 2 \end{bmatrix}$$

$$\begin{aligned} m^{e_1} &= \frac{\rho \cdot 2A \cdot L}{2 \cdot 6} \begin{bmatrix} 2 & 1 \\ 1 & 2 \end{bmatrix} \\ &= \frac{\rho A L}{12} \begin{bmatrix} 4 & 2 \\ 2 & 4 \end{bmatrix} \end{aligned}$$

$$m^{e_2} = \frac{\rho \cdot A L}{12} \begin{bmatrix} 2 & 1 \\ 1 & 2 \end{bmatrix}$$

Global Mass matrix

$$M^e = \frac{\rho A L}{12} \begin{bmatrix} 4 & 2 & 0 \\ 2 & 6 & 1 \\ 0 & 1 & 2 \end{bmatrix}$$

Eqn of motion

$$|K - \lambda M| = 0$$

$$\frac{EA}{L} \begin{bmatrix} 4 & -4 & 0 \\ -4 & 6 & -2 \\ 0 & -2 & 2 \end{bmatrix} - \lambda \begin{bmatrix} 4 & 2 & 0 \\ 2 & 6 & 1 \\ 0 & 1 & 2 \end{bmatrix} \frac{\rho A L}{12} = 0$$

$$\left| \begin{bmatrix} 4 & -4 & 0 \\ -4 & 6 & -2 \\ 0 & -2 & 2 \end{bmatrix} - \frac{\rho L^2}{12E} \lambda \begin{bmatrix} 4 & 2 & 0 \\ 2 & 6 & 1 \\ 0 & 1 & 2 \end{bmatrix} \right| = 0$$

$$\left| \begin{bmatrix} 6 & -2 \\ -2 & 2 \end{bmatrix} - b \begin{bmatrix} 6 & 1 \\ 1 & 2 \end{bmatrix} \right| = 0 \quad \therefore b = \frac{8L^2}{12E} \lambda$$

$$\begin{vmatrix} 6-6b & -(2+b) \\ -(2+b) & 2-2b \end{vmatrix} = 0$$

$$(6-6b)(2-2b) - (2+b)^2 = 0$$

$$12 - 12b - 12b + 12b^2 - 4 - b^2 - 4b = 0$$

$$8 - 28b + 11b^2 = 0$$

$$b_1 = 0.3279$$

$$b_2 = 2.217$$

$$\frac{8L^2}{12E} \lambda = 0.3279$$

$$\lambda_1 = \frac{3.9348 E}{8L^2} = \omega_1^2$$

$$\omega_1 = \frac{1.98}{L} \sqrt{\frac{E}{8}} \text{ rad/s}$$

$$\frac{8L^2}{12E} \lambda = 2.217$$

$$\lambda_2 = \frac{26.604 E}{8L^2} = \omega_2^2$$

$$\omega_2 = \frac{5.16}{L} \sqrt{\frac{E}{8}} \text{ rad/s}$$

For first mode shape

$$[K - \lambda M][x] = 0$$

$$\left(\frac{EA}{L} \begin{bmatrix} 6 & -2 \\ -2 & 2 \end{bmatrix} - \left(3.9348 \frac{E}{8L^2} \right) \cdot \frac{8AL}{12} \begin{bmatrix} 6 & 1 \\ 1 & 2 \end{bmatrix} \right) \begin{bmatrix} x_2^{(1)} \\ x_3^{(1)} \end{bmatrix} = \begin{bmatrix} 0 \\ 0 \end{bmatrix}$$

$$\left(\begin{bmatrix} 6 & -2 \\ -2 & 2 \end{bmatrix} - \begin{bmatrix} 1.98 & 0.33 \\ 0.33 & 0.66 \end{bmatrix} \right) \begin{bmatrix} x_2^{(1)} \\ x_3^{(1)} \end{bmatrix} = \begin{bmatrix} 0 \\ 0 \end{bmatrix}$$

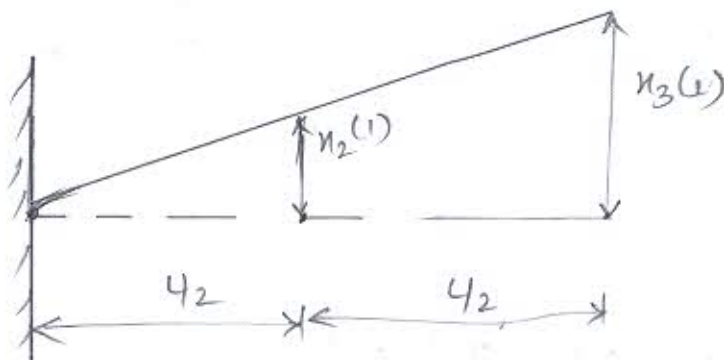
$$\begin{bmatrix} 4.02 & -2.33 \\ -2.33 & 1.34 \end{bmatrix} \begin{bmatrix} x_2^{(1)} \\ x_3^{(1)} \end{bmatrix} = \begin{bmatrix} 0 \\ 0 \end{bmatrix}$$

$$4.02 x_2^{(1)} - 2.33 x_3^{(1)} = 0$$

$$x_3^{(1)} = 1.72 x_2^{(1)}$$

Eigen vector or 1st mode shape

$$\begin{bmatrix} x^{(1)} \end{bmatrix} = \begin{bmatrix} x_2^{(1)} \\ x_3^{(1)} \end{bmatrix} = \begin{bmatrix} x_2^{(1)} \\ 1.72 x_2^{(1)} \end{bmatrix} = x_2^{(1)} \begin{bmatrix} 1 \\ 1.72 \end{bmatrix}$$



For Second mode shape

put $\omega_2^2 = \lambda_2$ in $(K - \lambda M) [x] = 0$.

$$\left(\frac{EA}{L} \begin{bmatrix} 6 & -2 \\ -2 & 2 \end{bmatrix} - \left(\frac{26.604 E}{8L^2} \right) \begin{bmatrix} 6 & 1 \\ 1 & 2 \end{bmatrix} \frac{8AK}{12} \right) \begin{bmatrix} x_2^{(2)} \\ x_3^{(2)} \end{bmatrix} = \begin{bmatrix} 0 \\ 0 \end{bmatrix}$$

$$\left(\begin{bmatrix} 6 & -2 \\ -2 & 2 \end{bmatrix} - 2.217 \begin{bmatrix} 6 & 1 \\ 1 & 2 \end{bmatrix} \right) \begin{bmatrix} x_2^{(2)} \\ x_3^{(2)} \end{bmatrix} = \begin{bmatrix} 0 \\ 0 \end{bmatrix}$$

$$\left(\begin{bmatrix} 6 & -2 \\ -2 & 2 \end{bmatrix} - \begin{bmatrix} 13.302 & 2.217 \\ 2.217 & 4.434 \end{bmatrix} \right) \begin{bmatrix} x_2^{(2)} \\ x_3^{(2)} \end{bmatrix} = \begin{bmatrix} 0 \\ 0 \end{bmatrix}$$

$$\begin{bmatrix} 6 - 13.302 & -2 - 2.217 \\ -2 - 2.217 & 2 - 4.434 \end{bmatrix} \begin{bmatrix} x_2^{(2)} \\ x_3^{(2)} \end{bmatrix} = \begin{bmatrix} 0 \\ 0 \end{bmatrix}$$

$$\begin{bmatrix} -7.302 & -4.217 \\ -4.217 & -2.434 \end{bmatrix} \begin{bmatrix} x_2^{(2)} \\ x_3^{(2)} \end{bmatrix} = \begin{bmatrix} 0 \\ 0 \end{bmatrix}$$

$$-7.302 x_2^{(2)} - 4.217 x_3^{(2)} = 0$$

$$x_3^{(2)} = -1.72 x_2^{(2)}$$

$$\begin{bmatrix} x^{(2)} \end{bmatrix} = \begin{bmatrix} x_2^{(2)} \\ x_3^{(2)} \end{bmatrix} = \begin{bmatrix} x_2^{(2)} \\ -1.72 x_3^{(2)} \end{bmatrix}$$

$$\approx x_2^{(2)} \begin{bmatrix} 1 \\ -1.72 \end{bmatrix} //$$

